

Modelling, Poles and Zero's, Stepresponses

Job van Amerongen

Control Laboratory, Department of Electrical Engineering
University of Twente, Netherlands

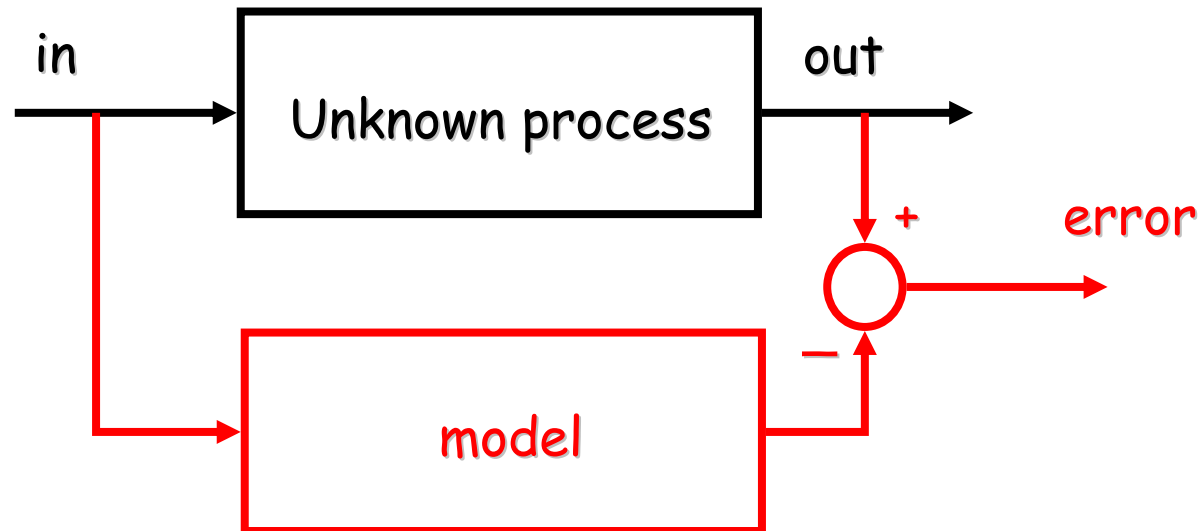
www.ce.utwente.nl/amn
J.vanAmerongen@utwente.nl

- **Modelling**
 - Physical modelling
 - Black box
 - Grey box
- **Descriptions**
 - IPM's
 - Bond graph
 - Block diagram
 - Frequency characteristic
 - State space
 - Time response

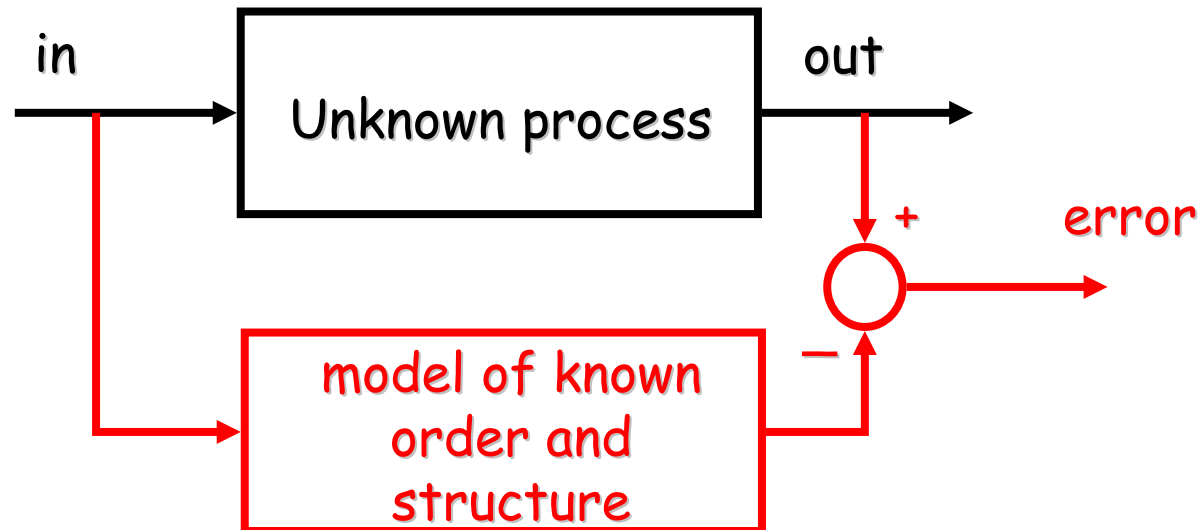
- Relations poles and zero's and step responses.
 - first order systems
 - second order systems
 - influence of zero's
 - initial and final value theorems.
- Relation between (step) responses and poles and zero's (**identification**)

- if a clear idea of the process and the various physical parts is available
- during the design of a (control) system
 - based on physical laws and geometrical properties (dynamische systemen)

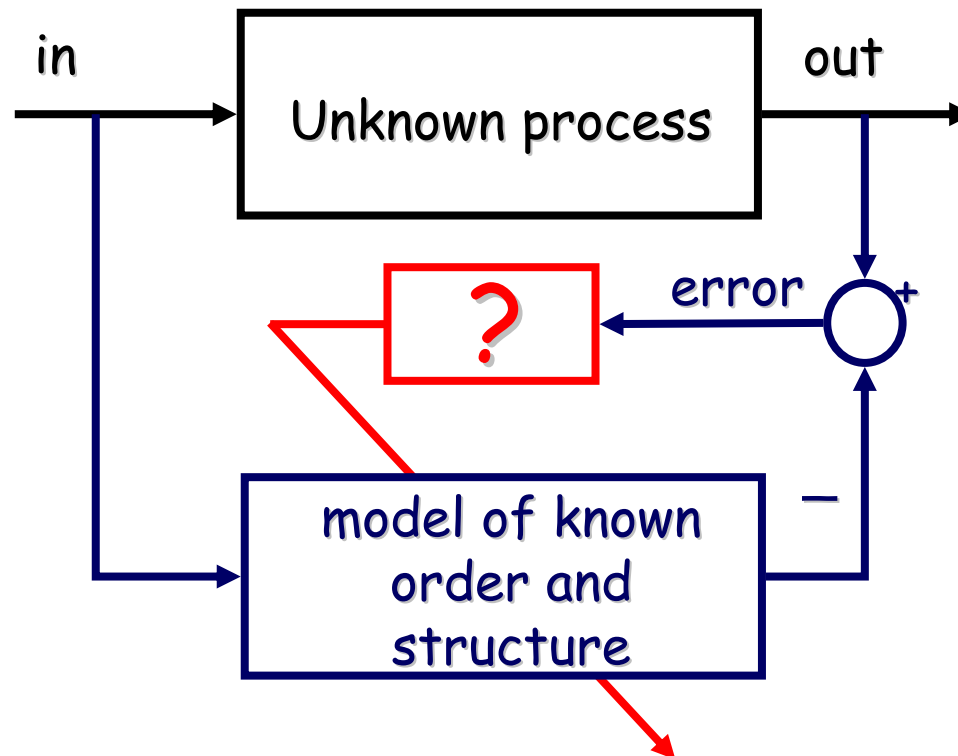
- When only measurements are available



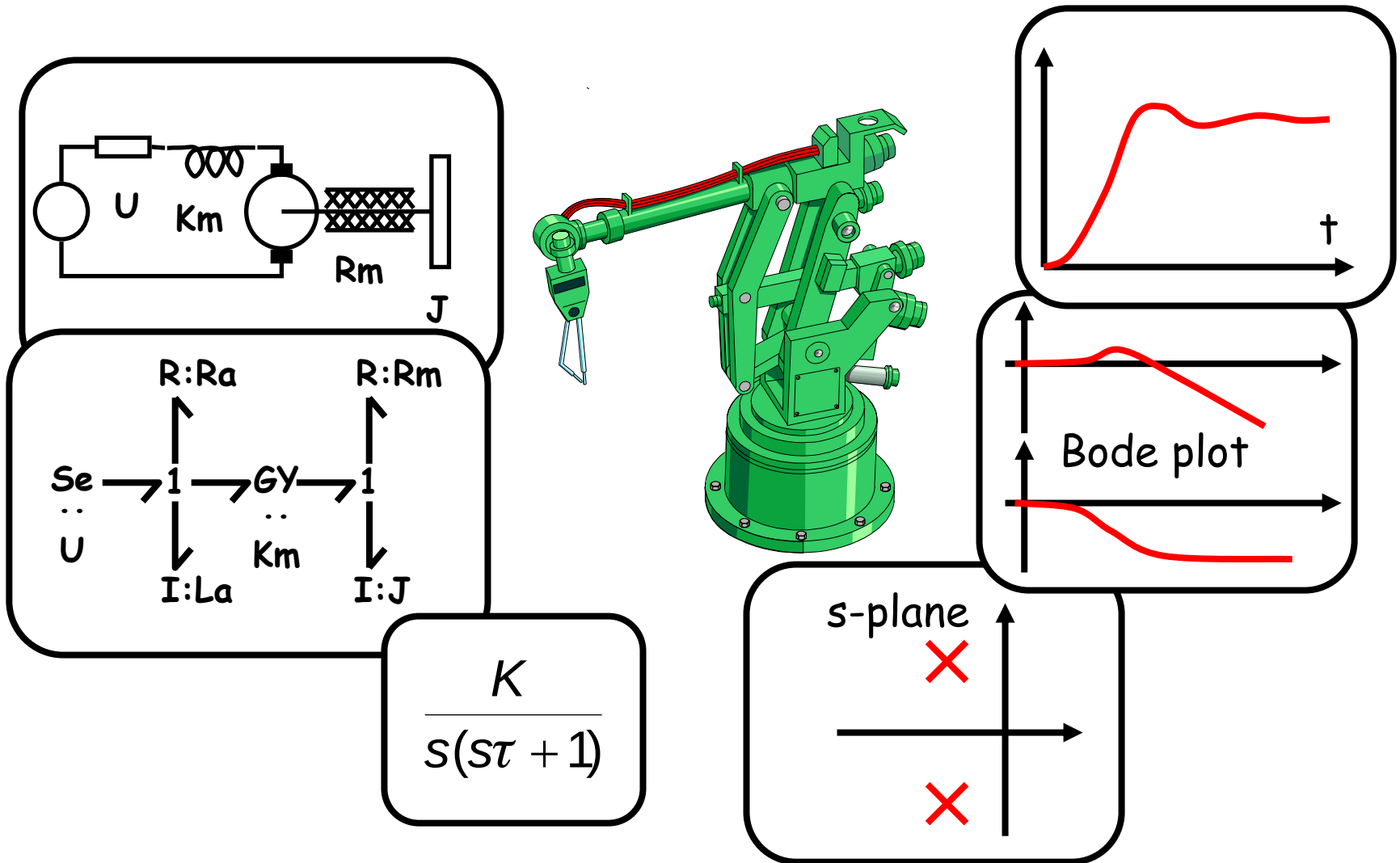
- Process structure and order is (assumed to be) known

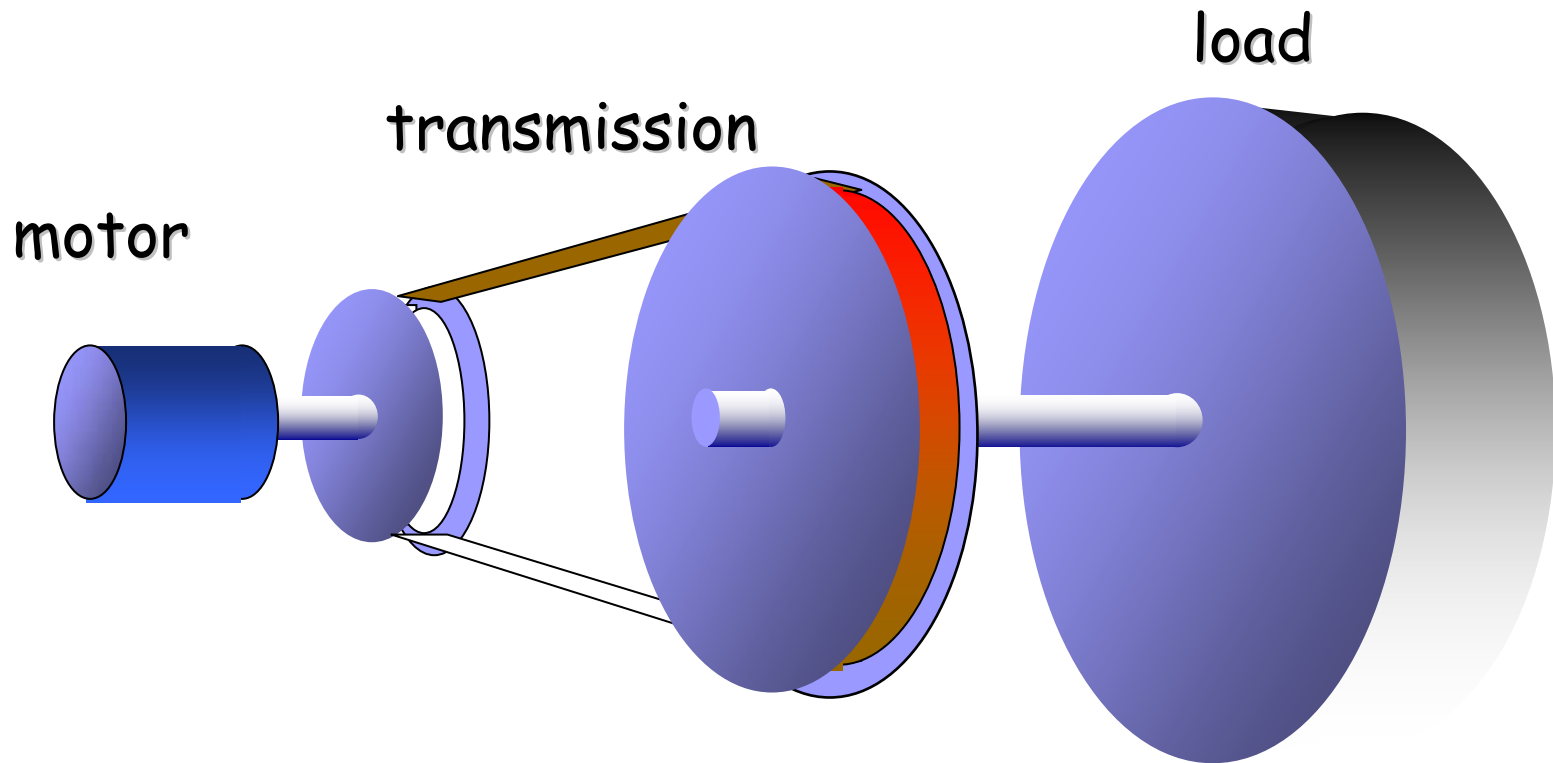


- Process structure and order is (assumed to be) known

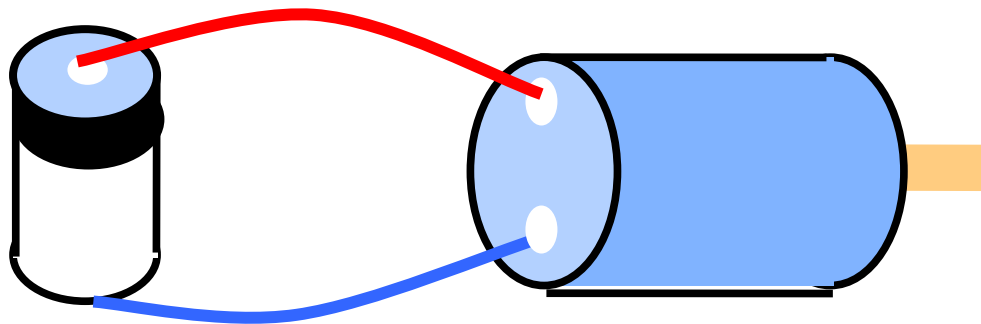


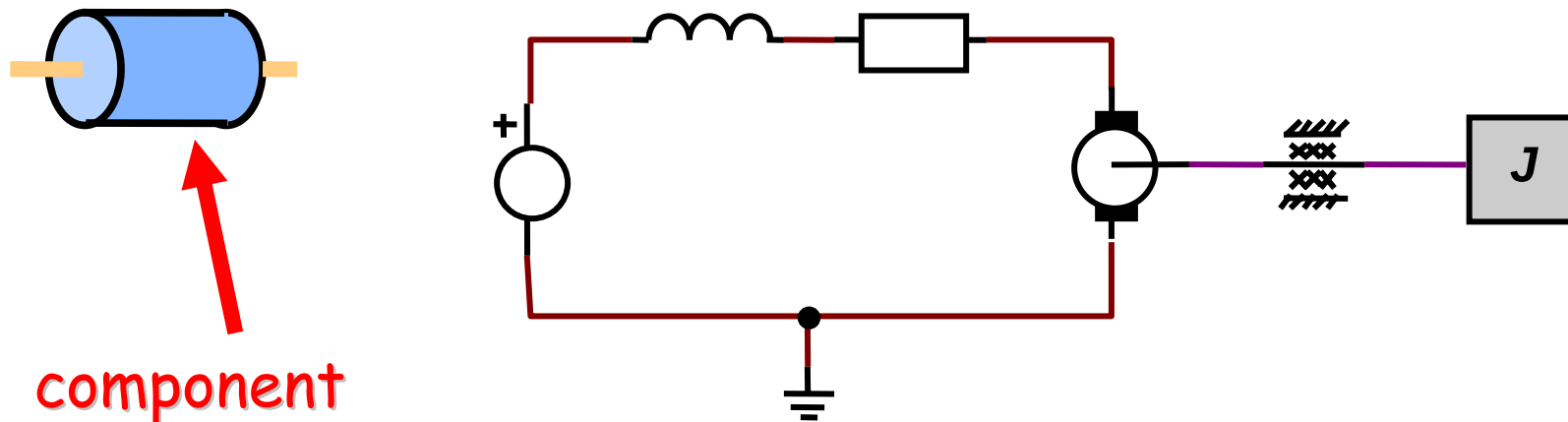
Multiple views





- Starts with considering the components in the process
- Model the process with a sufficiently level of detail
- i.e make a competent model





Ideal elements

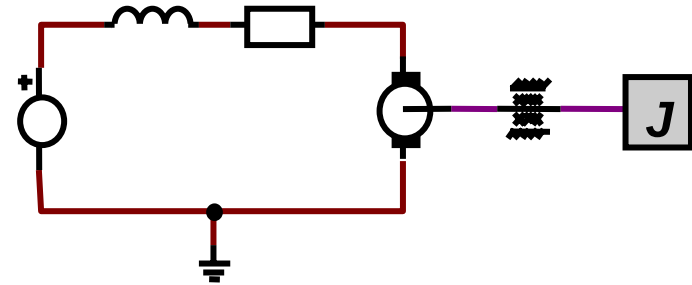
$$U - L_a \frac{di}{dt} - K_m \omega - R_a i_a = 0$$

$$K_m i - J \frac{d\omega}{dt} - R_m \omega = 0$$



$$L_a \frac{di}{dt} + R_a i_a = U - K_m \omega$$

$$J \frac{d\omega}{dt} + R_m \omega = K_m i$$



$$i_a = \frac{1}{L_a} \int (U - K_m \omega - R_a i_a) dt$$

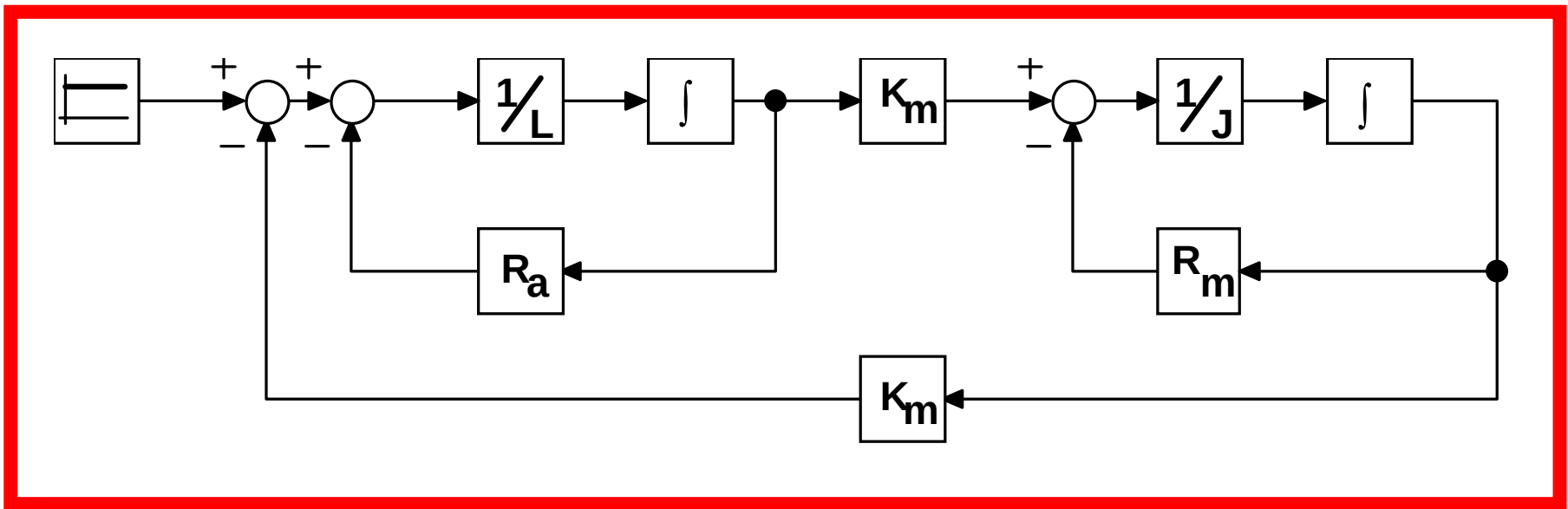
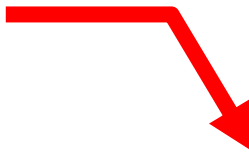


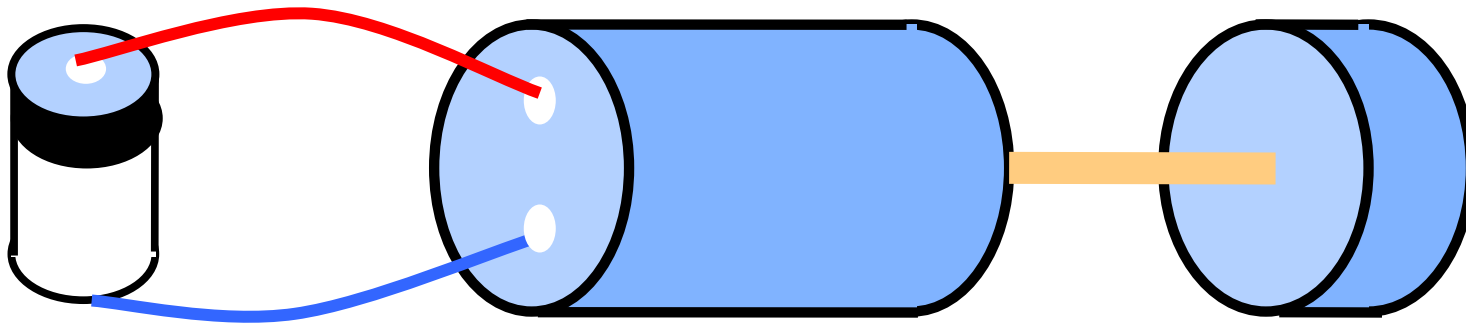
$$\omega = \frac{1}{J} \int (K_m i - R_m \omega) dt$$

equations → block diagram

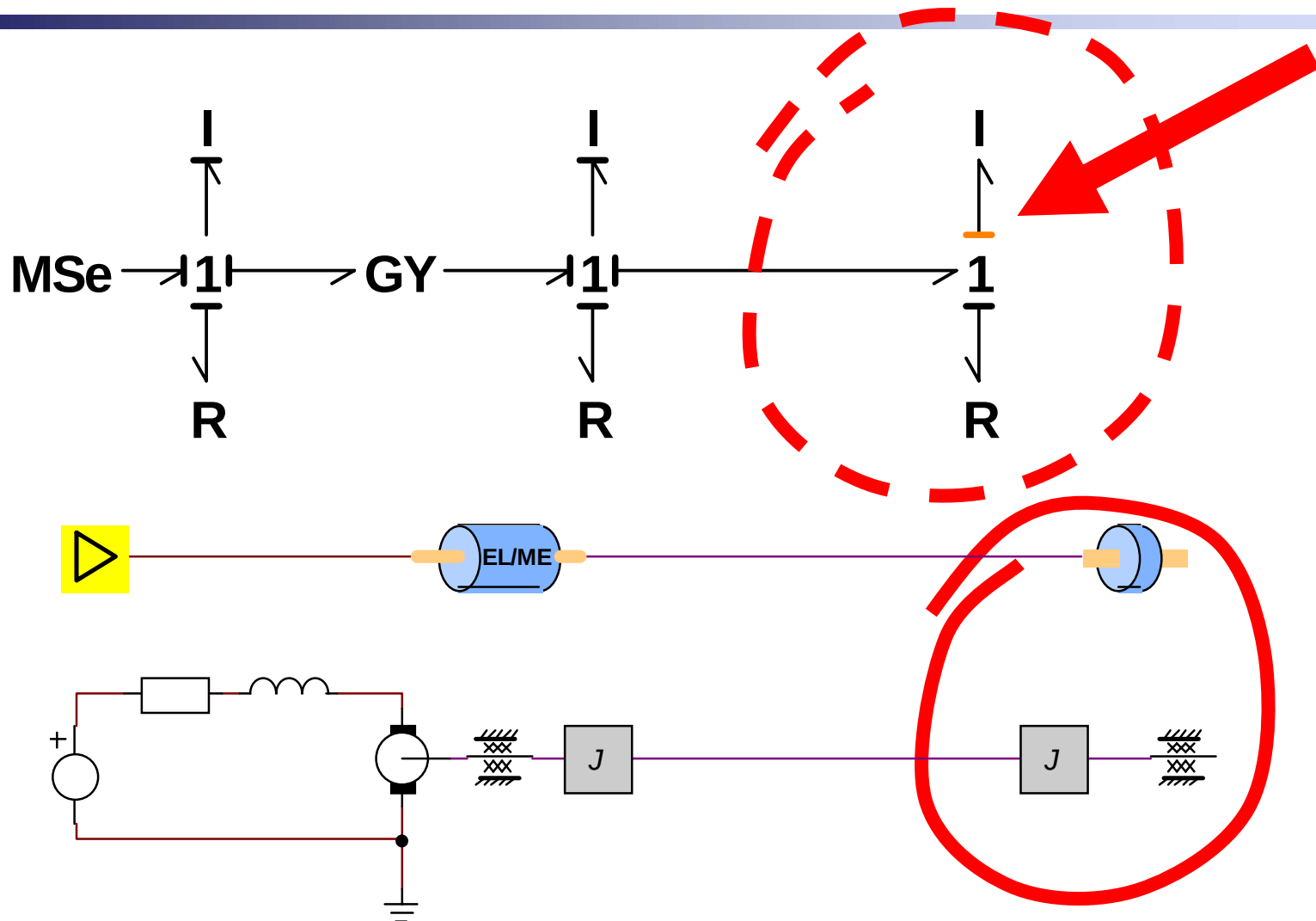
$$i_a = \frac{1}{L_a} \int (U - K_m \omega - R_a i_a) dt$$

$$\omega = \frac{1}{J} \int (K_m i - R_m \omega) dt$$

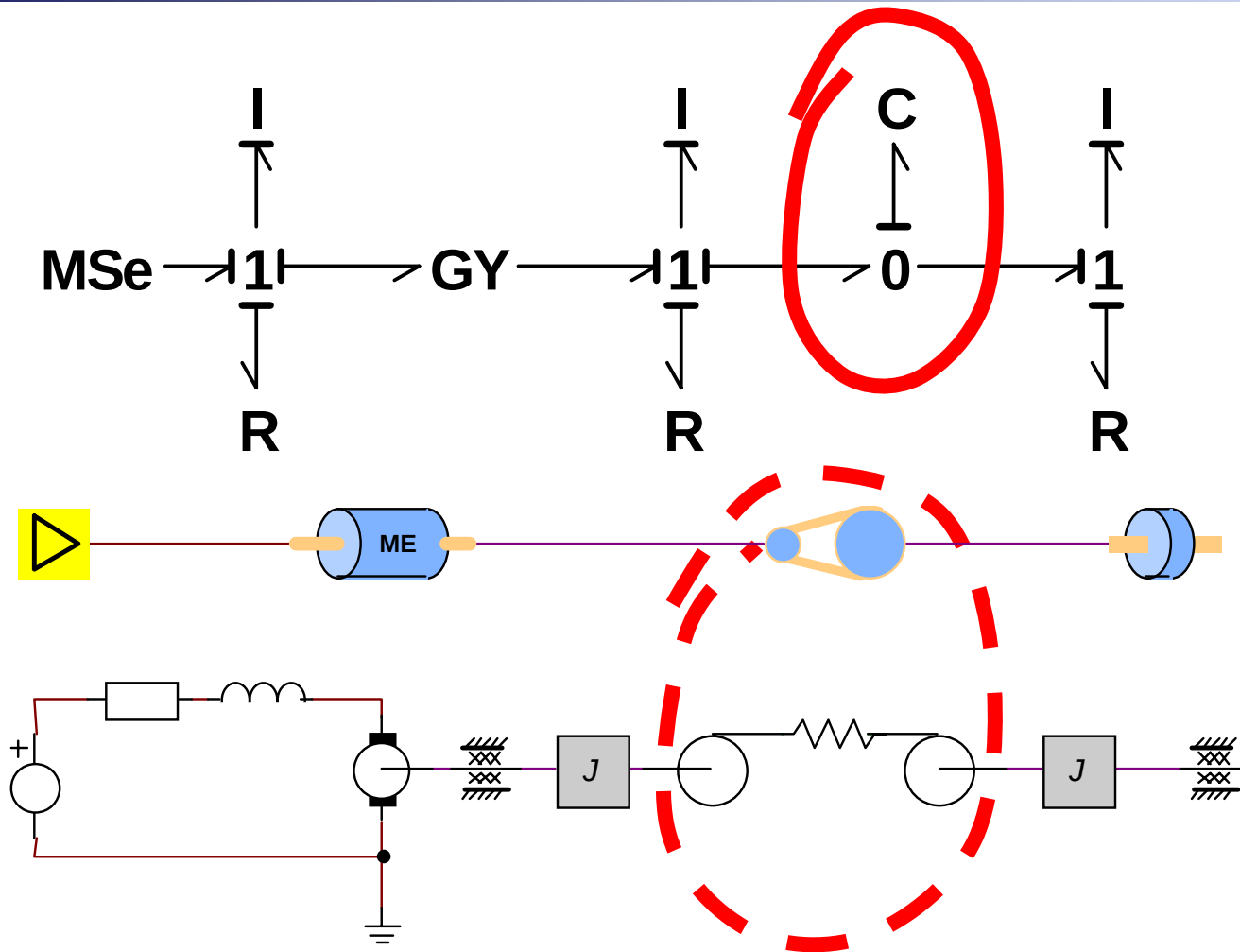




Add the load



Add flexible axis



$$U - L_a \frac{di}{dt} - K_m \omega_1 - R_a i_a = 0$$

$$K_m i_a - J_1 \frac{d\omega_1}{dt} - R_m \omega_1 = 0$$

$$\frac{1}{C} \left(\int (\omega_1 - \omega_2) dt \right) - T_{spring} = 0$$



$$i_a = \frac{1}{L_a} \left(\int (U - K_m \omega_1 - R_a i_a) dt \right)$$

$$\omega_1 = \frac{1}{J_1} \left(\int (K_m i_a - R_m \omega_1 - T_{spring}) dt \right)$$

$$T_{spring} = \frac{1}{C} \left(\int (\omega_1 - \omega_2) dt \right)$$

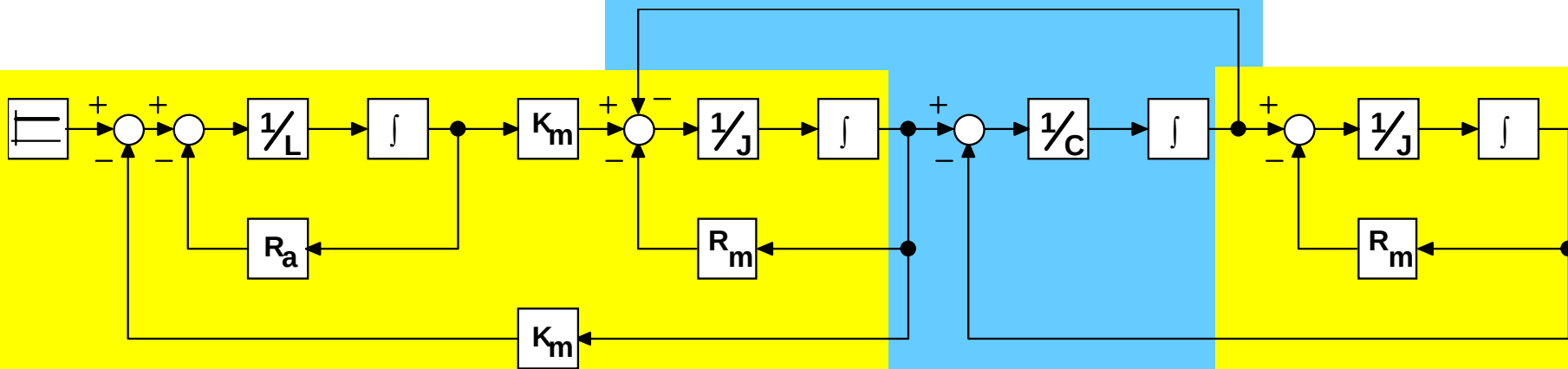
$$\omega_2 = \frac{1}{J_2} \left(\int (T_{spring} - R_m \omega_2) dt \right)$$

$$i_a = \frac{1}{L_a} \left(\int (U - K_m \omega_1 - R_a i_a) dt \right)$$

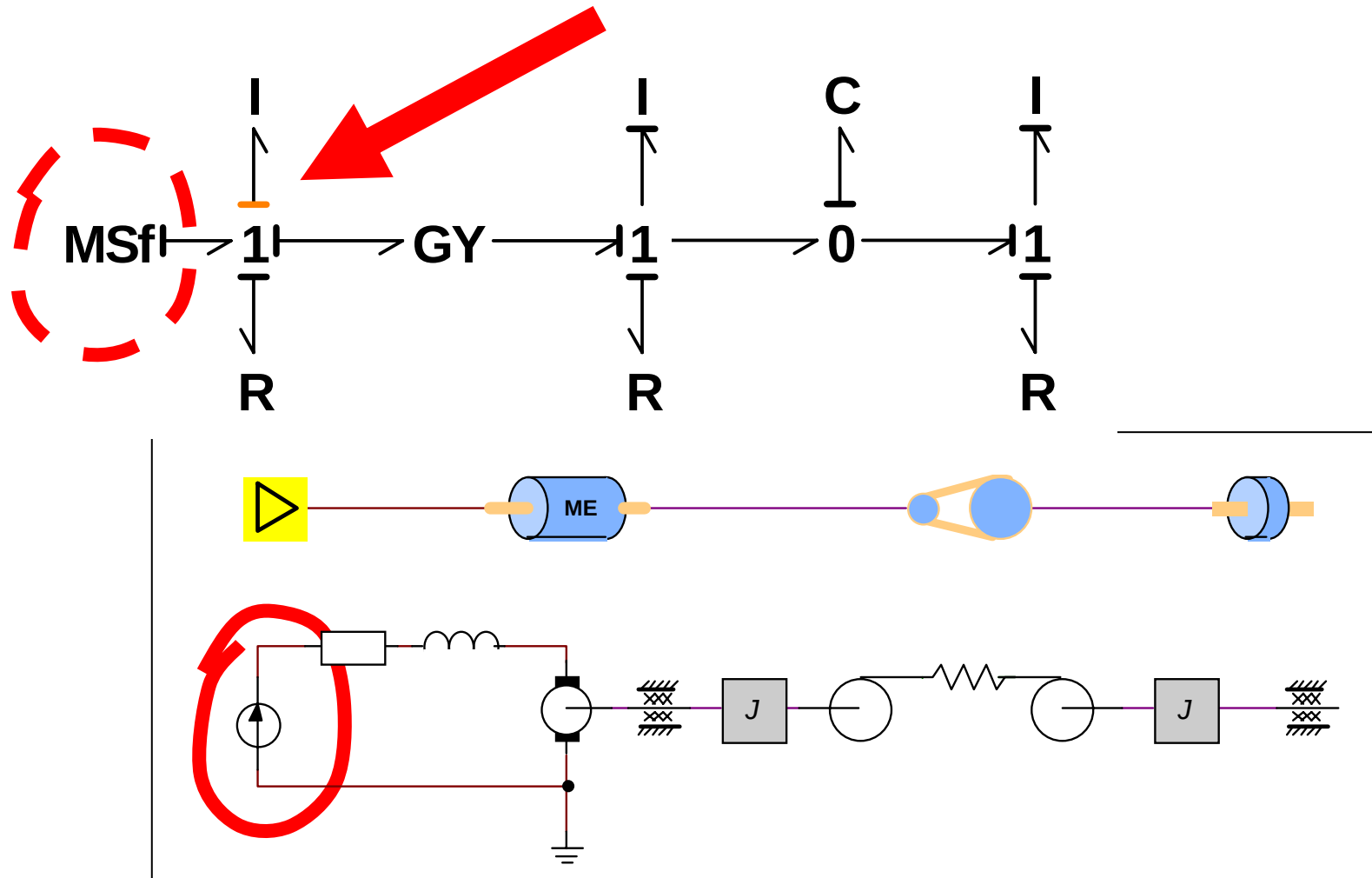
$$\omega_1 = \frac{1}{J_1} \left(\int (K_m i_a - R_m \omega_1 - T_{spring}) dt \right)$$

$$T_{spring} = \frac{1}{C} \left(\int (\omega_1 - \omega_2) dt \right)$$

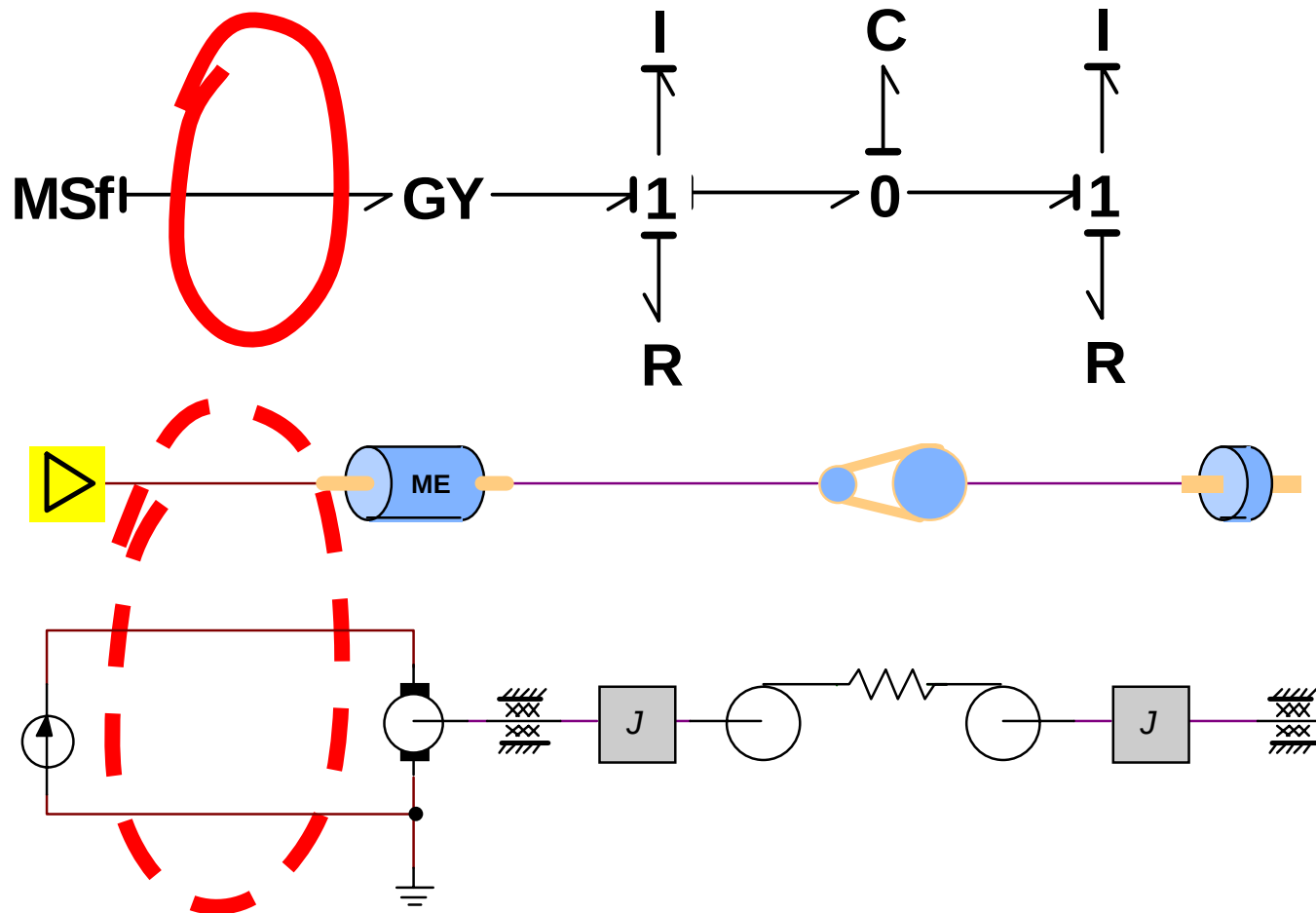
$$\omega_2 = \frac{1}{J_2} \left(\int (T_{spring} - R_m \omega_2) dt \right)$$

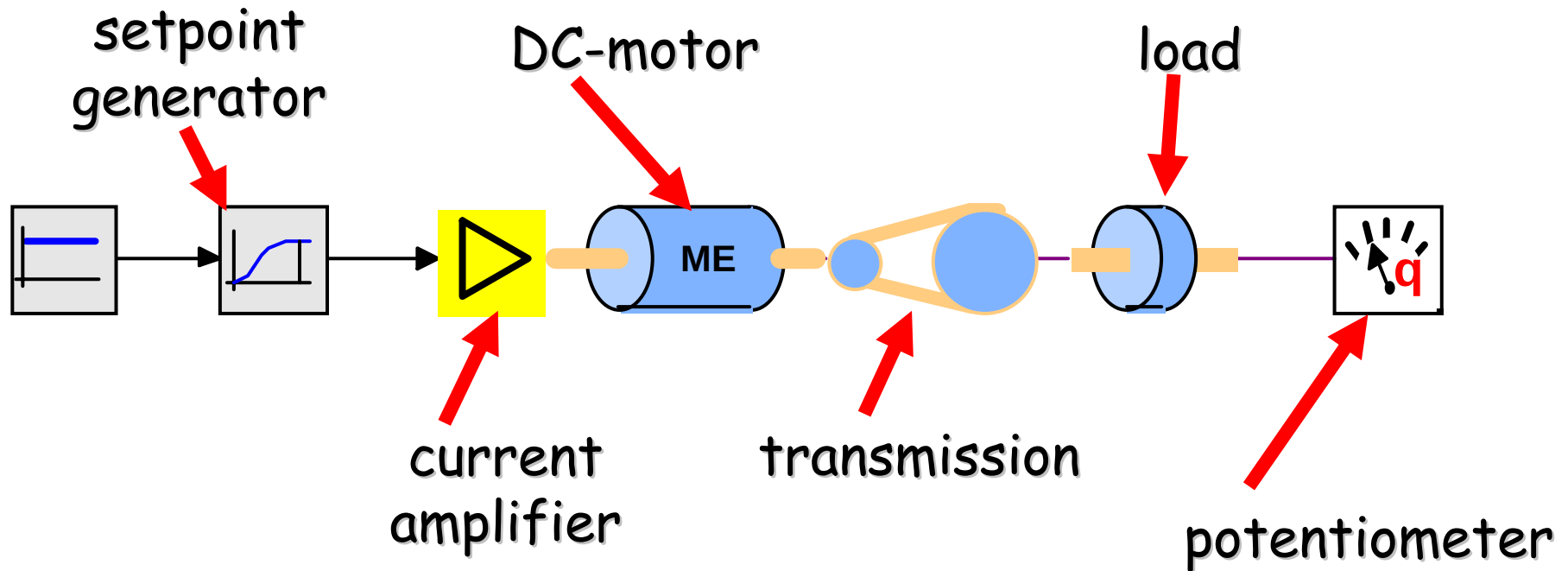


Use current amplifier



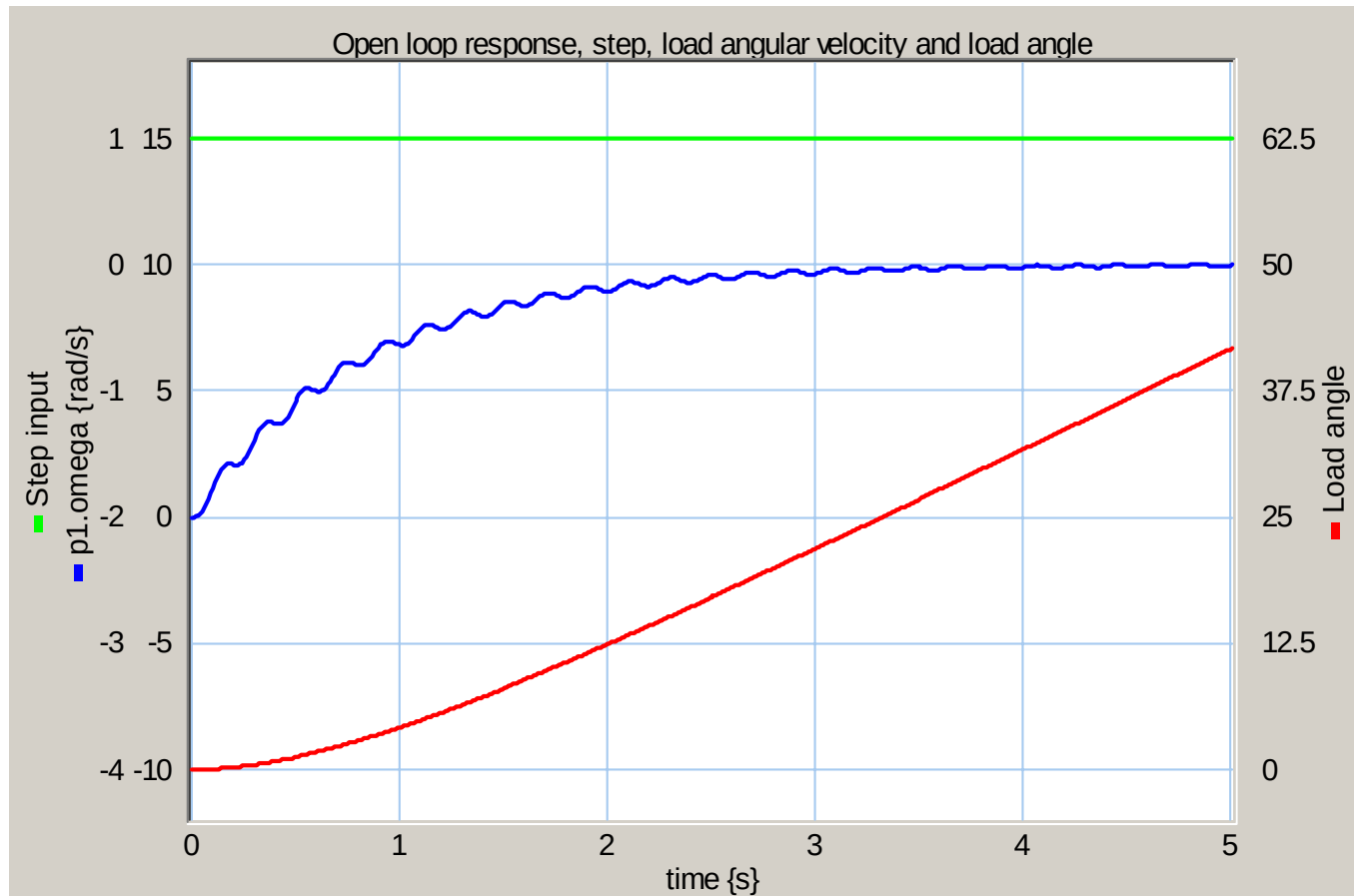
No electrical time constant



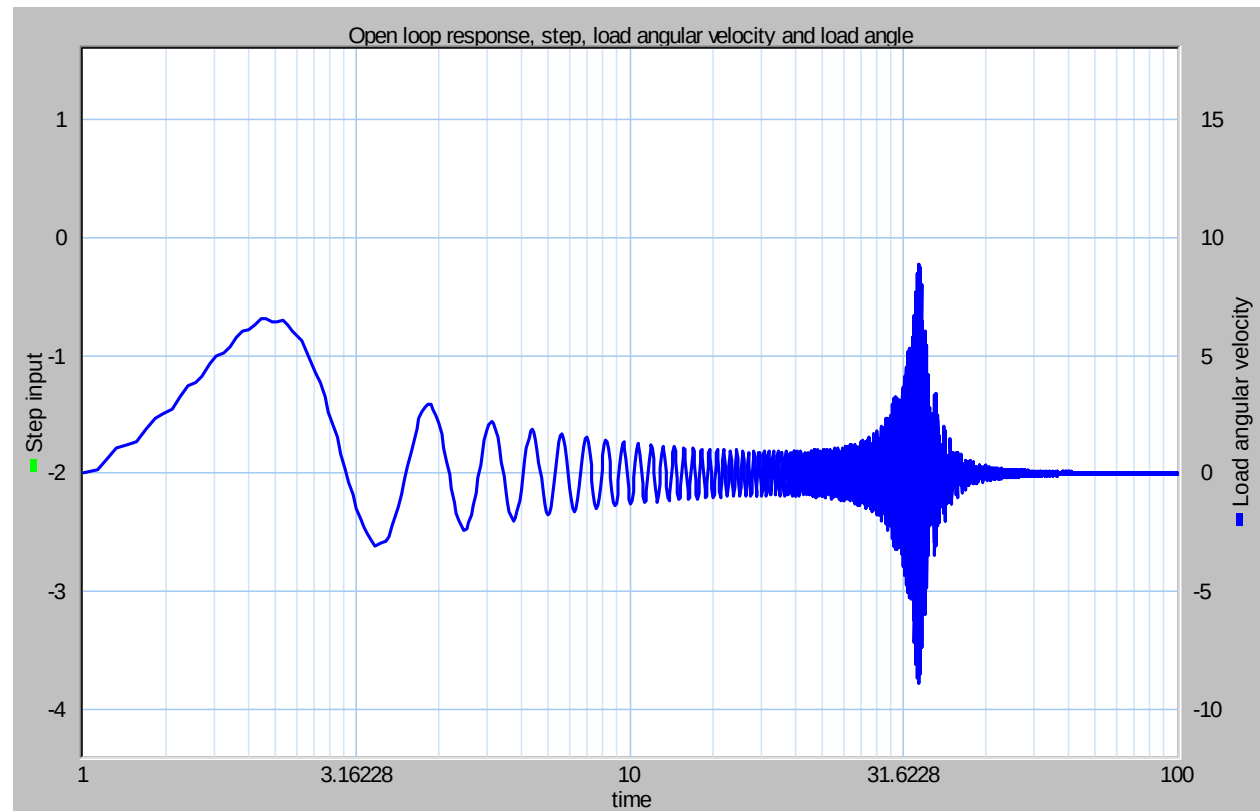


20-sim
demo

Step response

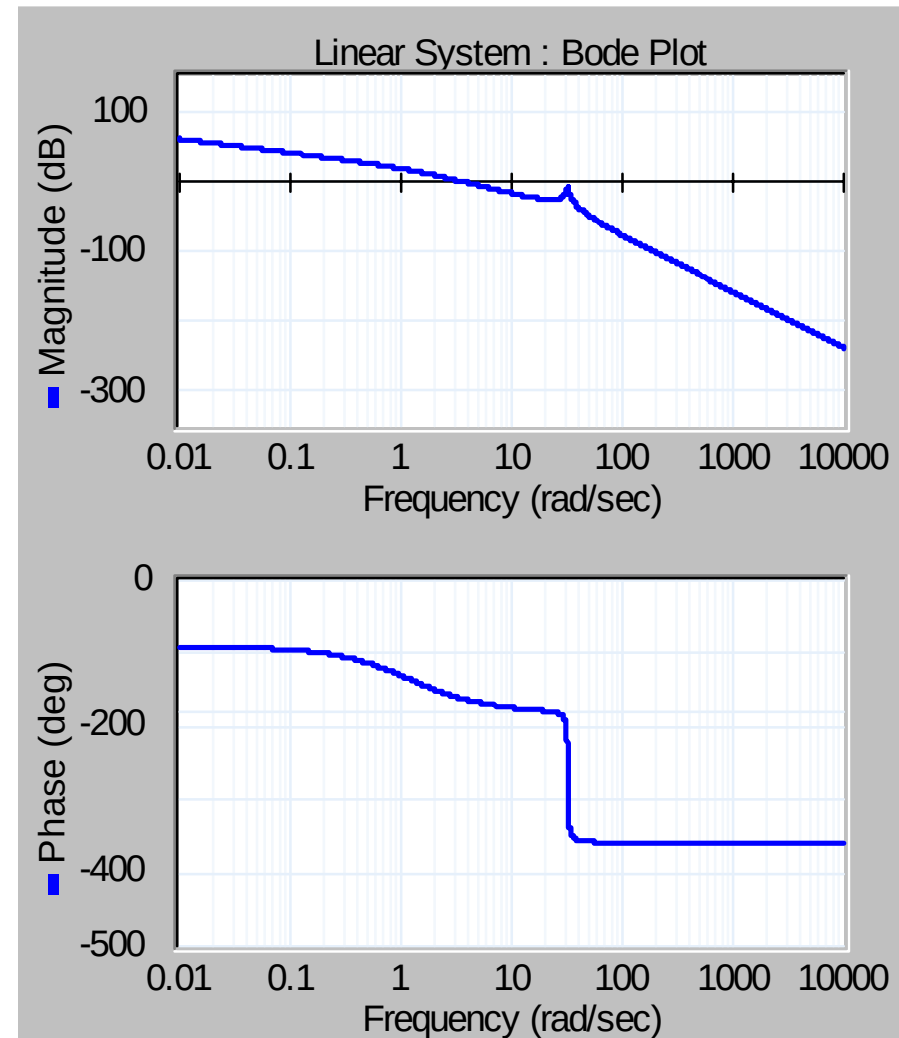
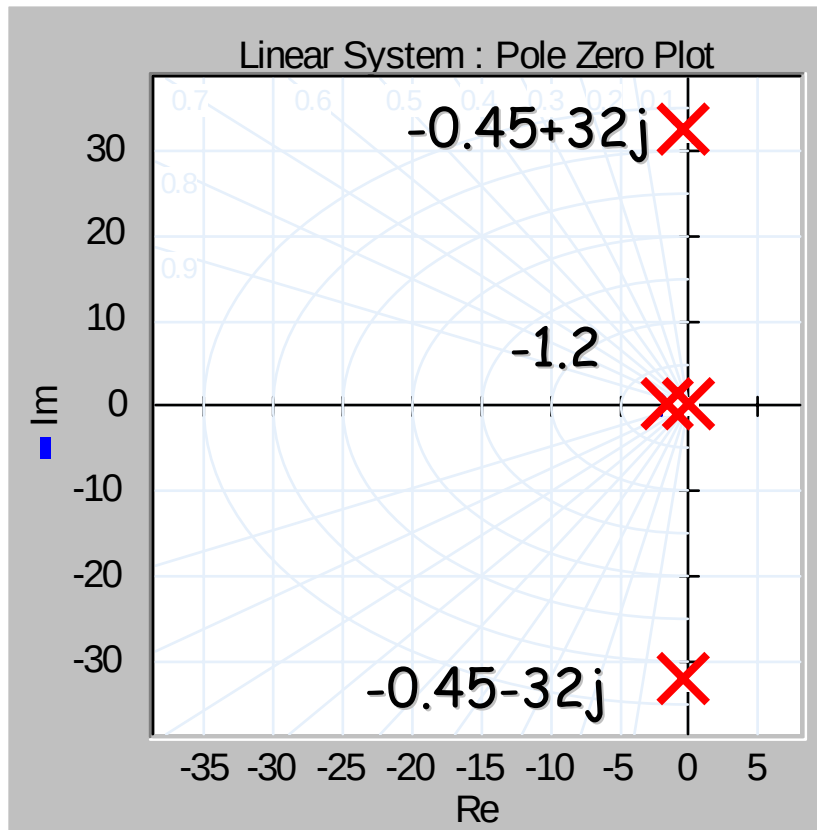


Response to a sweep

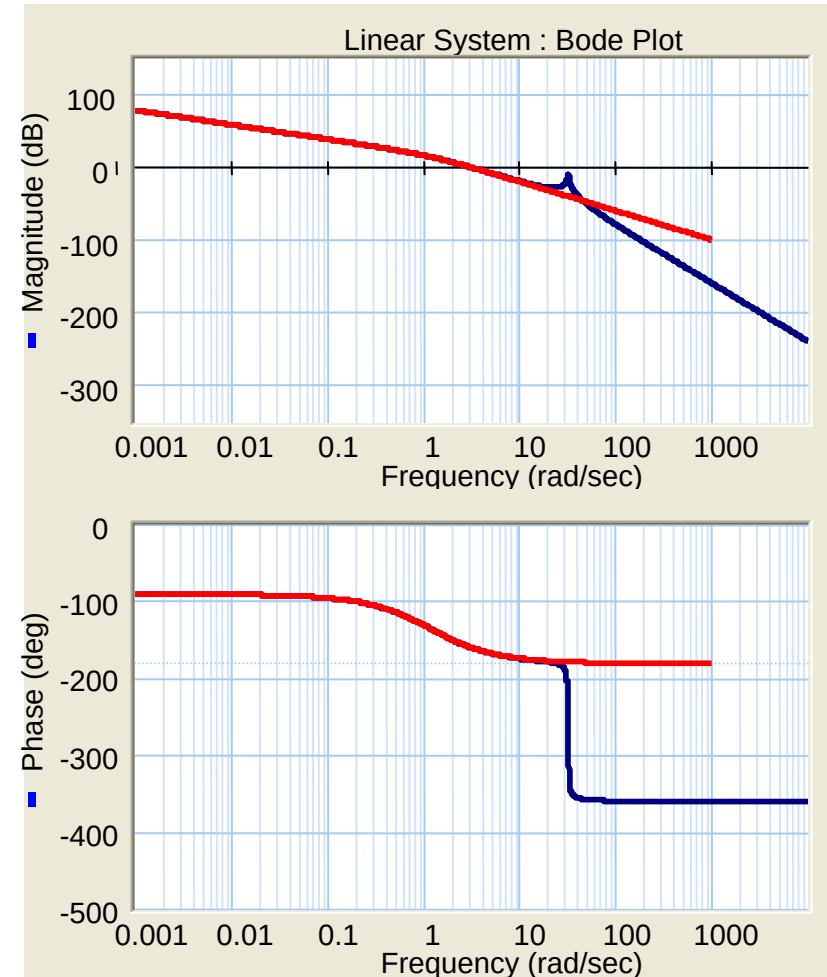
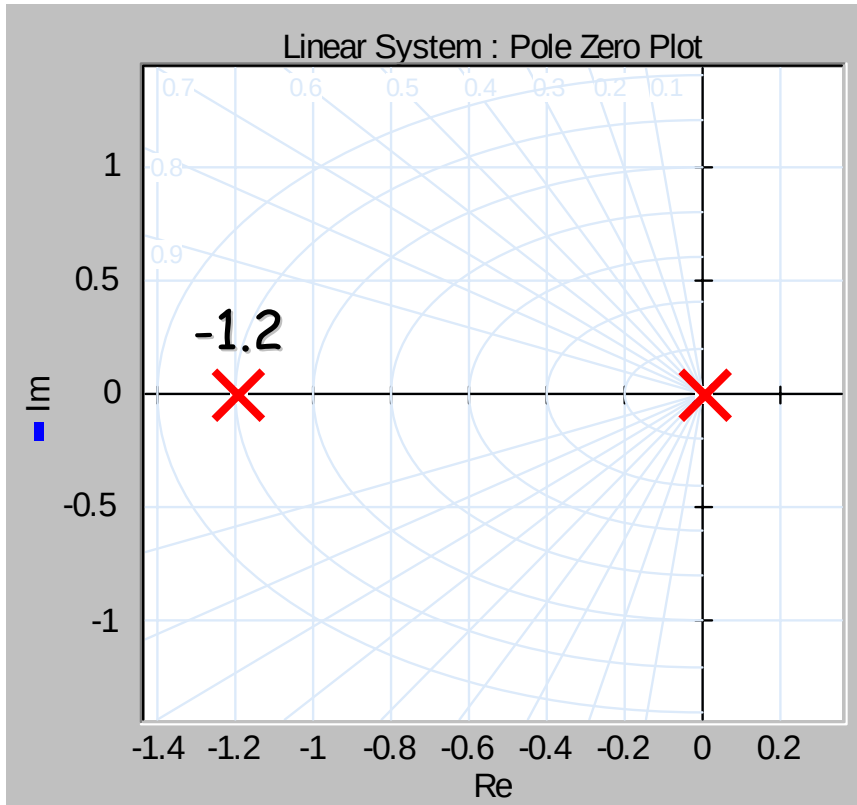


20-sim
demo

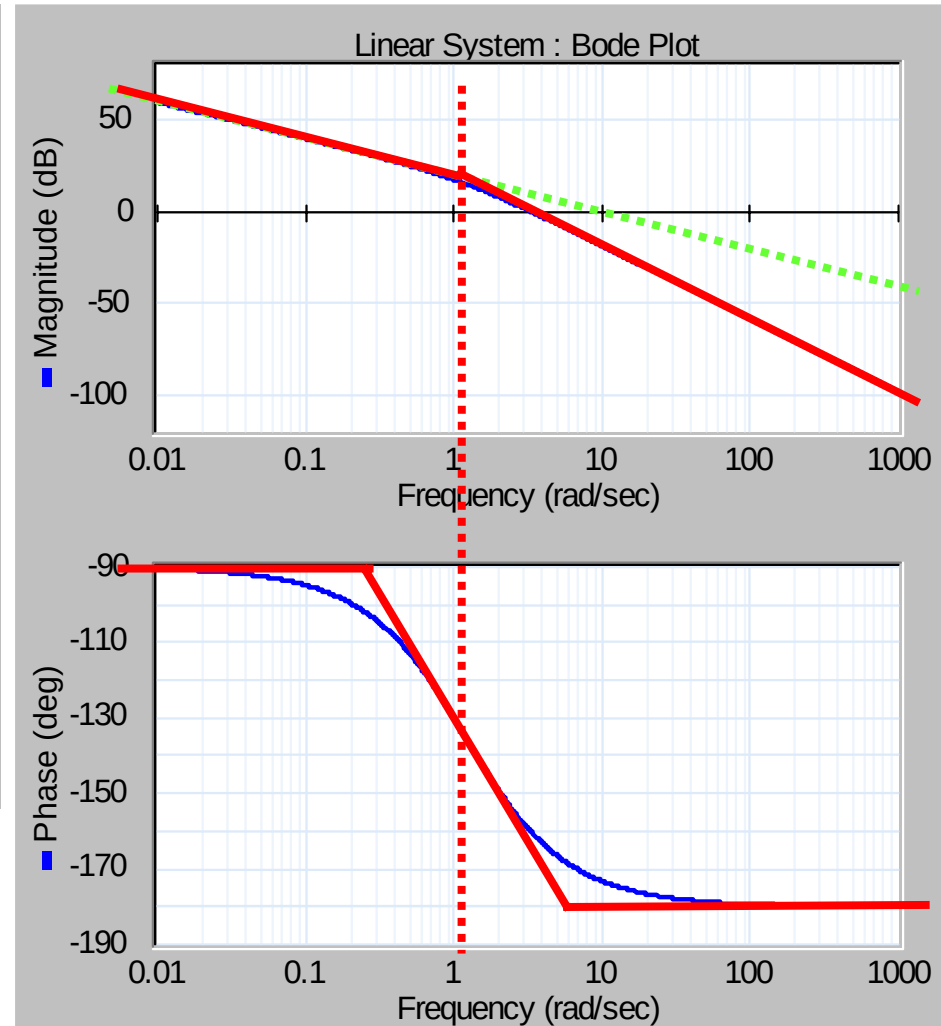
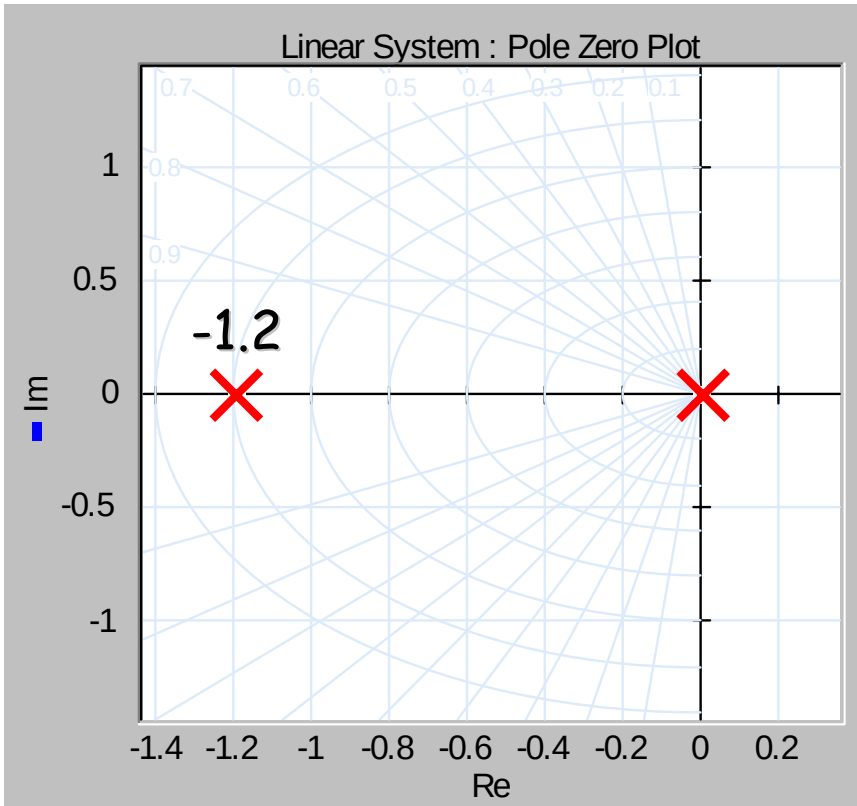
After 'linearisation' (current)



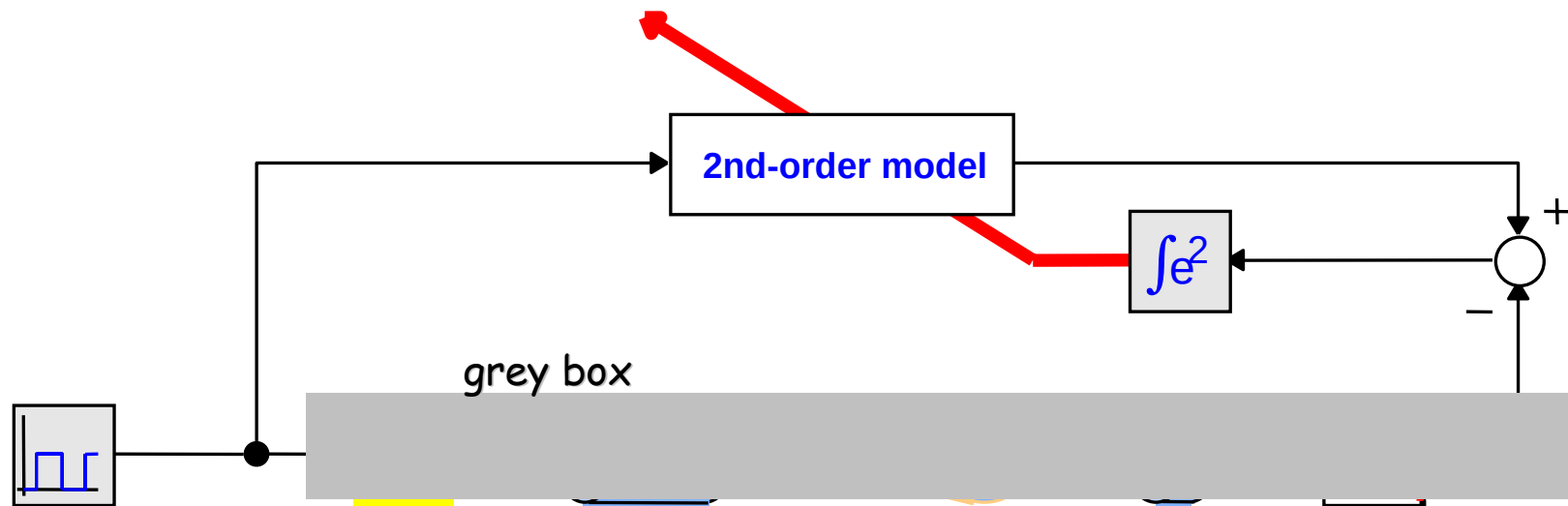
After simplification (current)



After simplification (current)



- Starts with measurements of an existing system
- Fits a (simple) dynamic model through the response

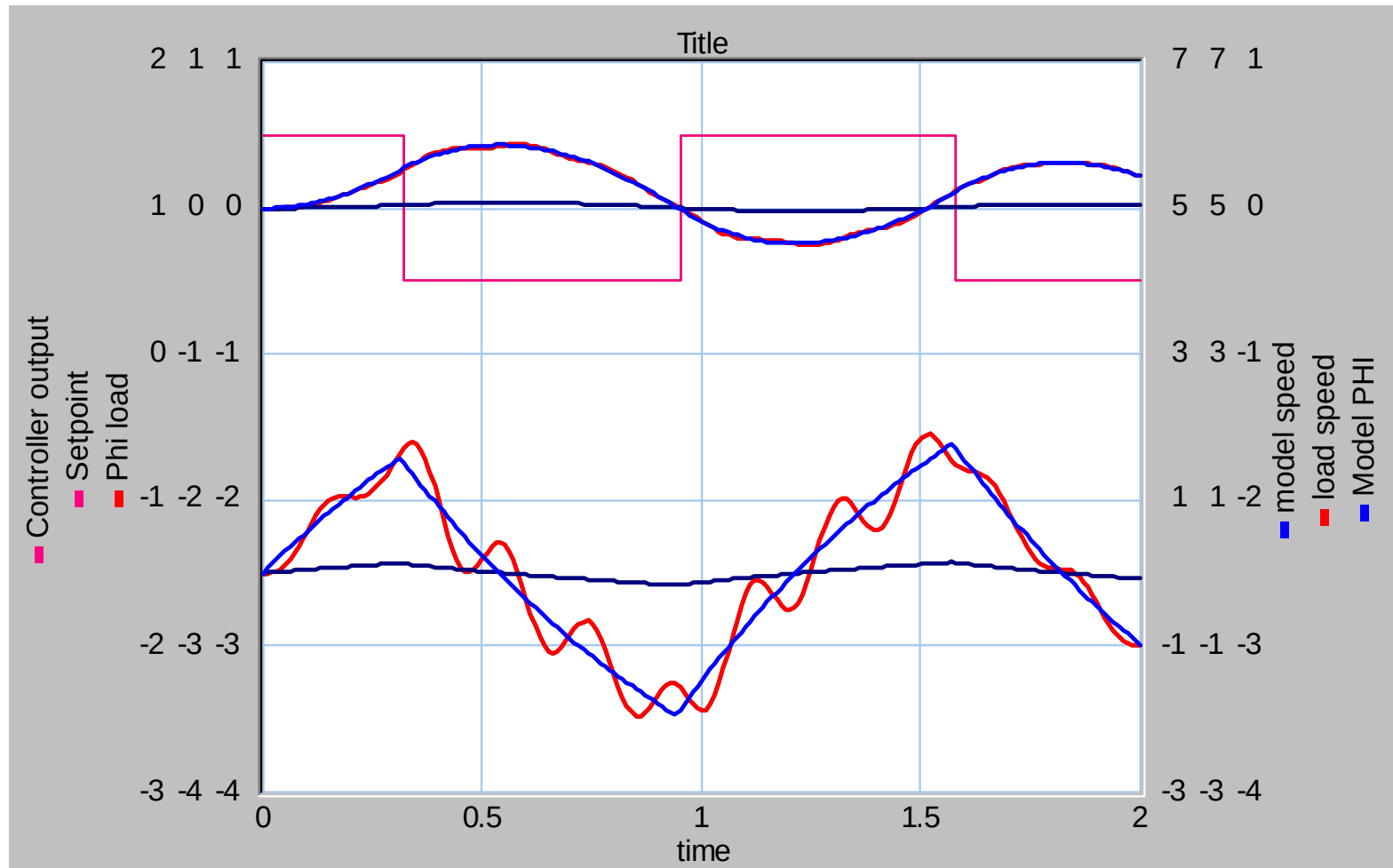


This model demonstrates how a 4th-order system can be approximated by a 2nd-order system

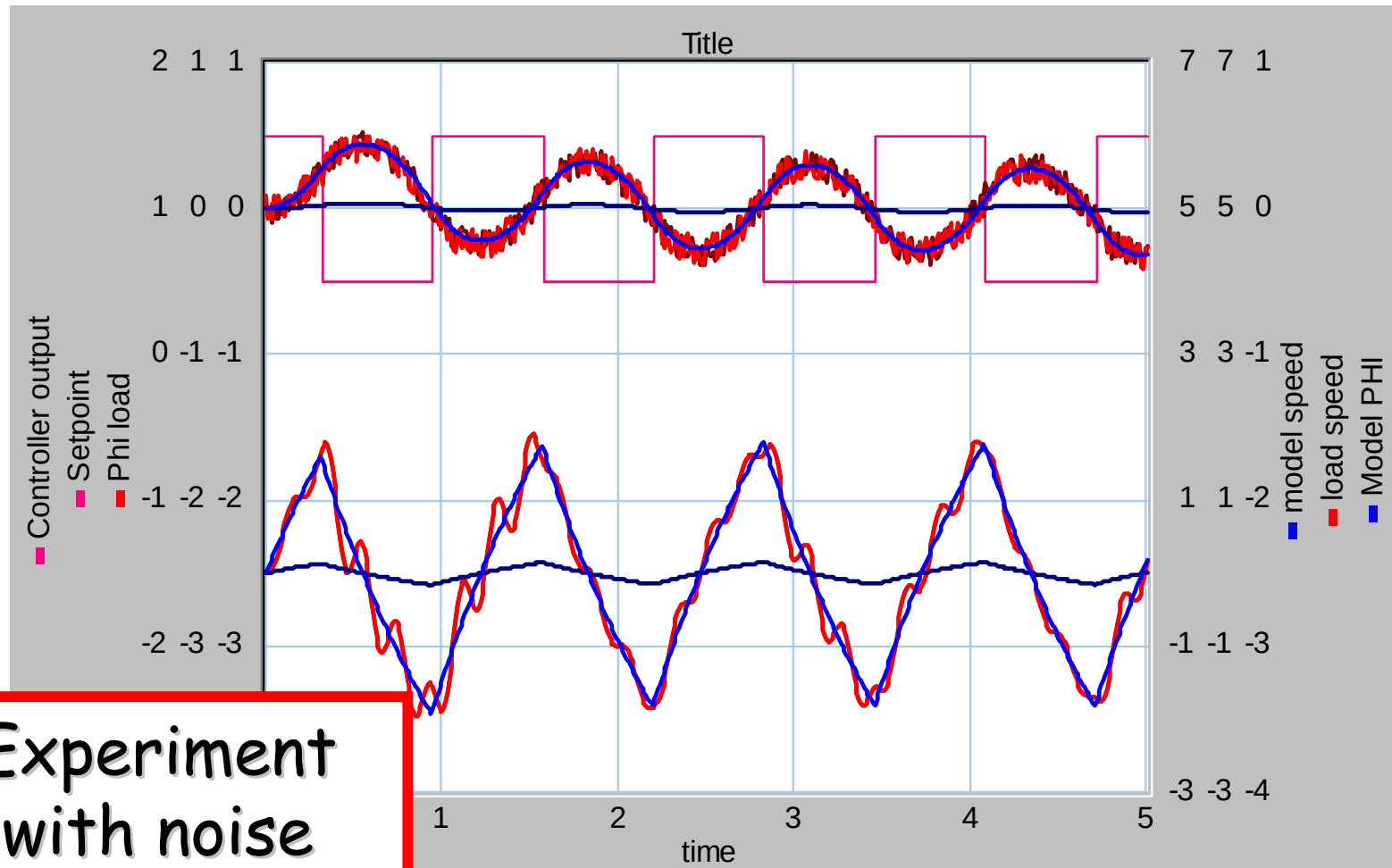
Run the experiment for identification

20-sim

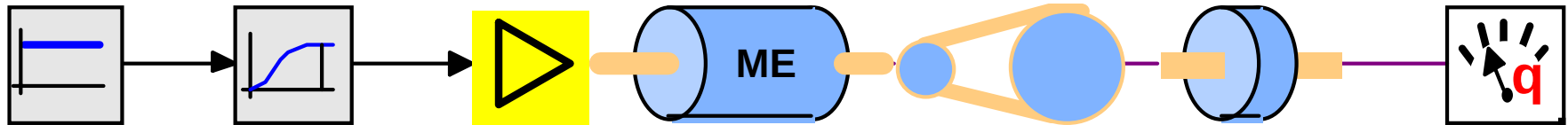
Hill climbing



Hill climbing (noise)



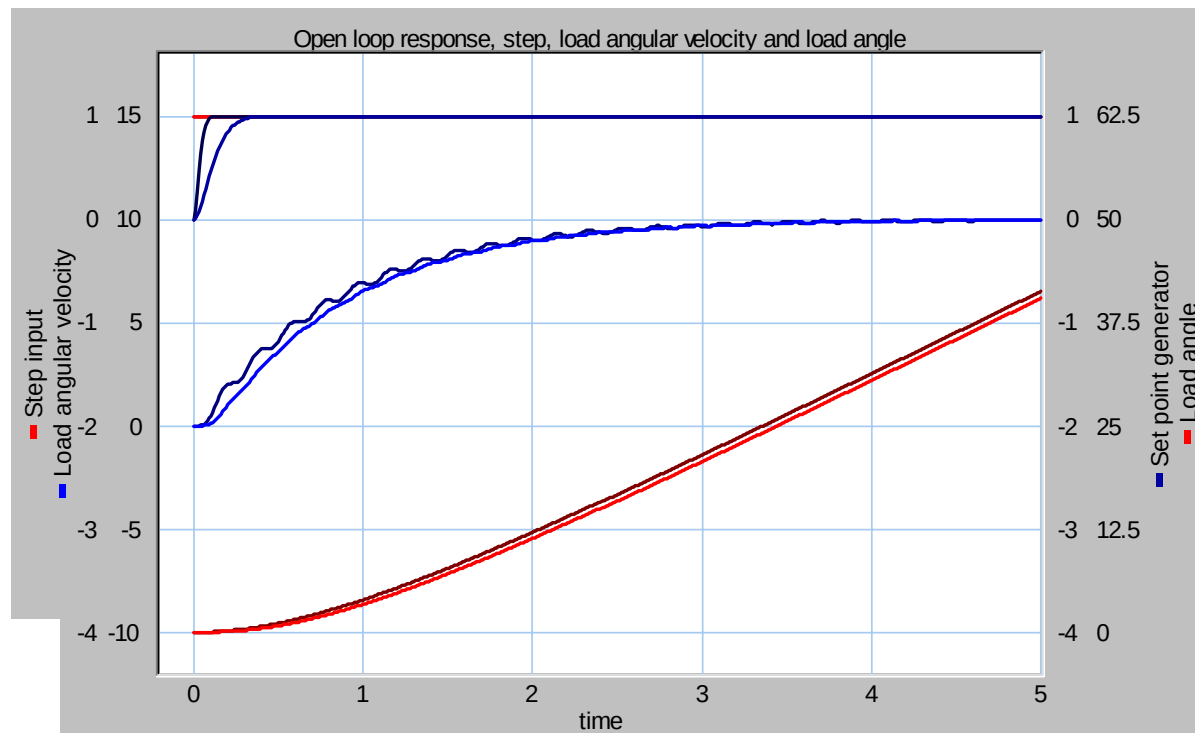
Experiment
with noise



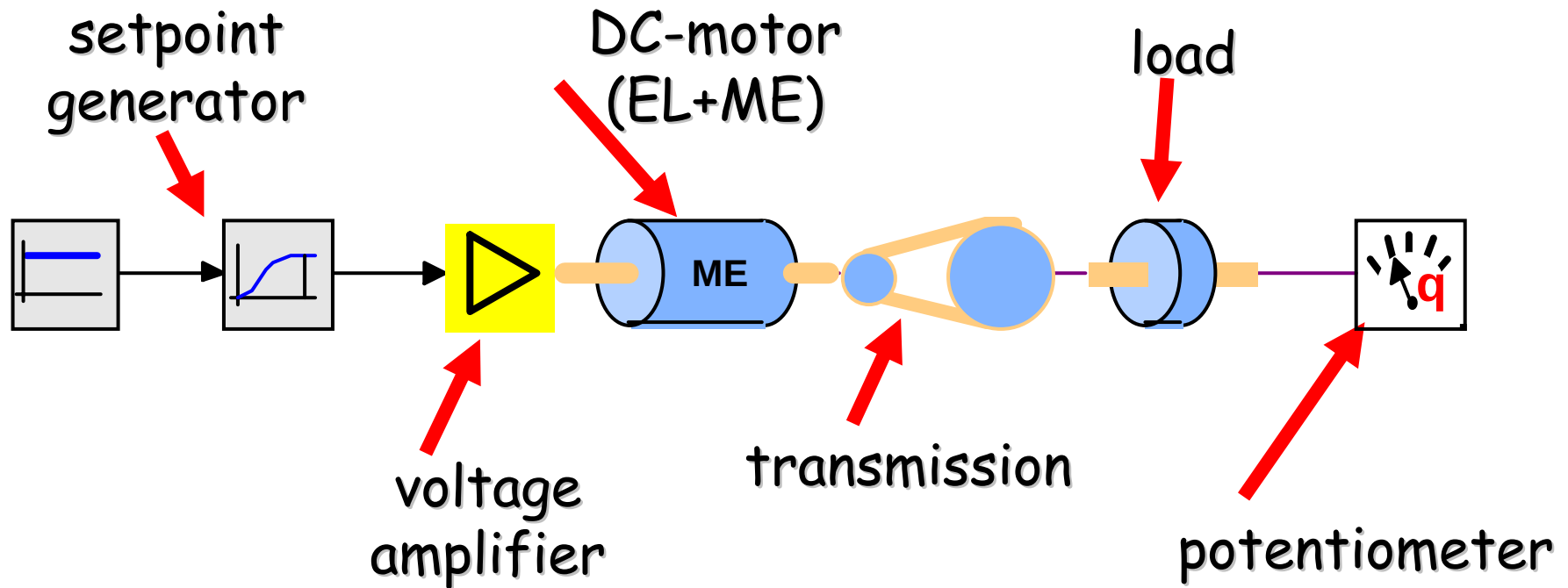
$\omega = 50$

$\omega = 15$

demo
20-sim



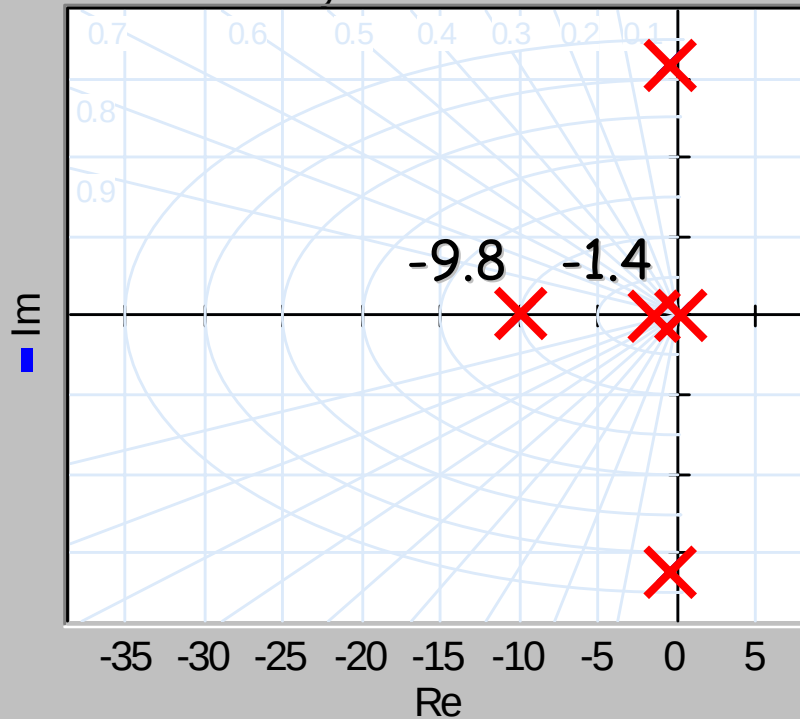
Demo process (volt)



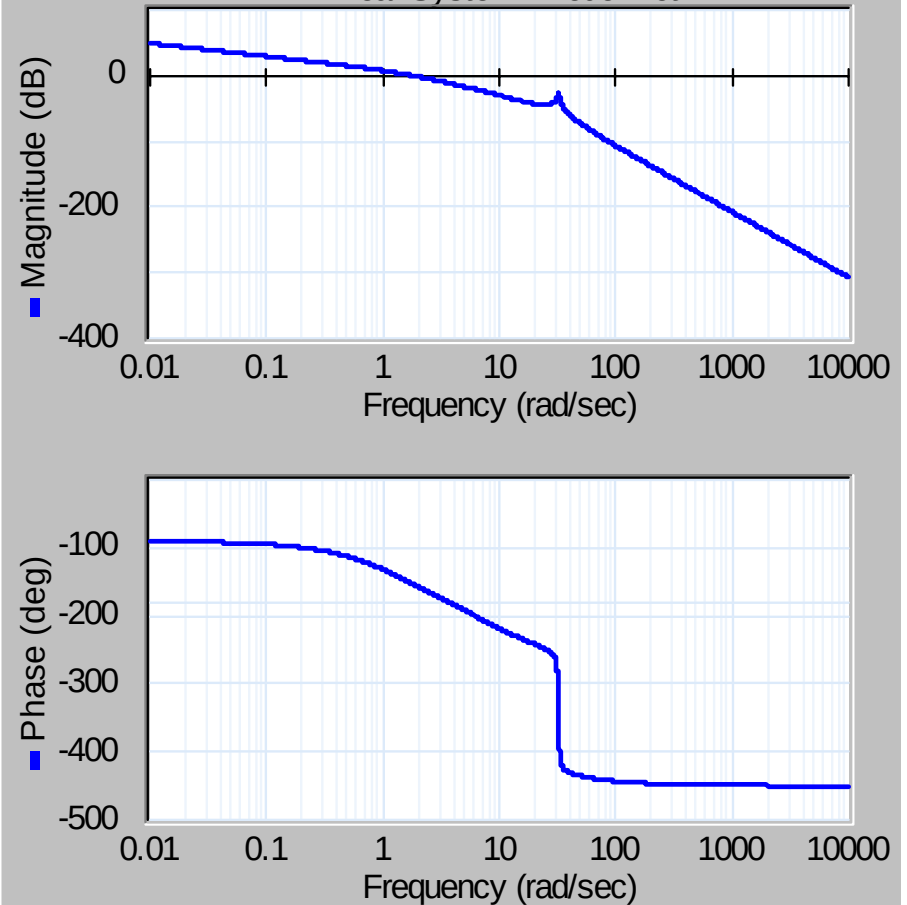
Demo
20-sim

After 'linearisation' (volt.)

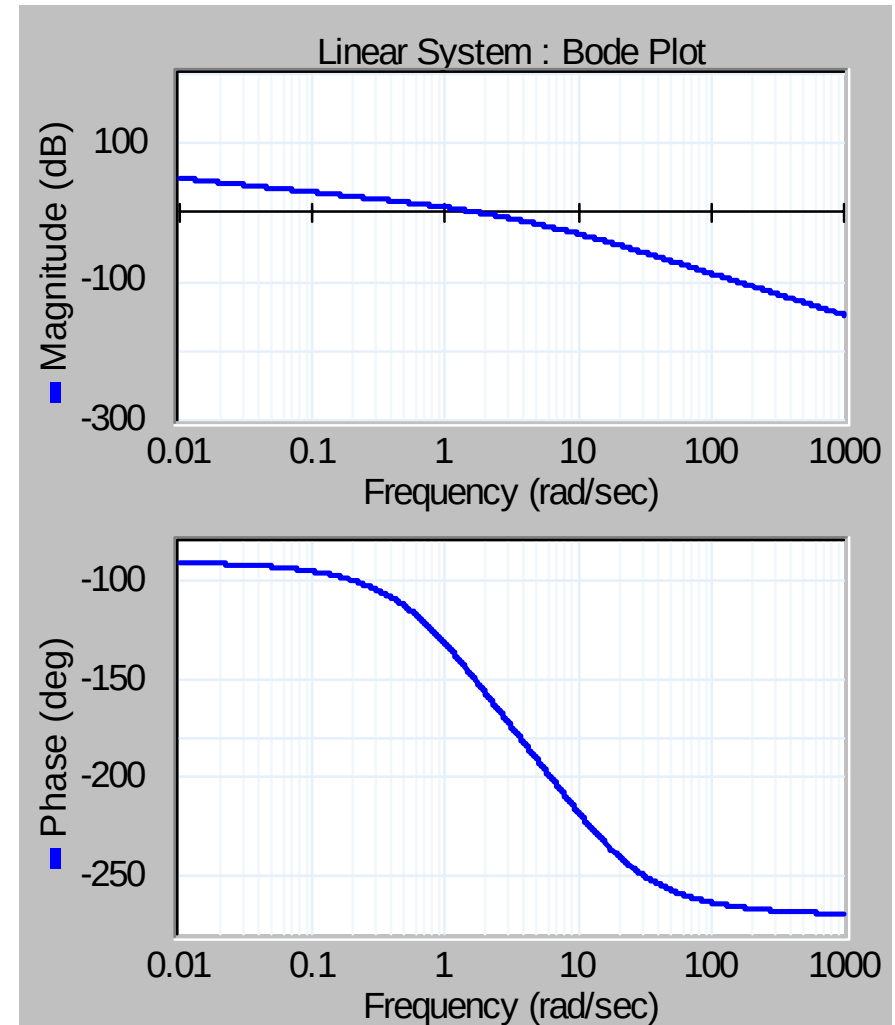
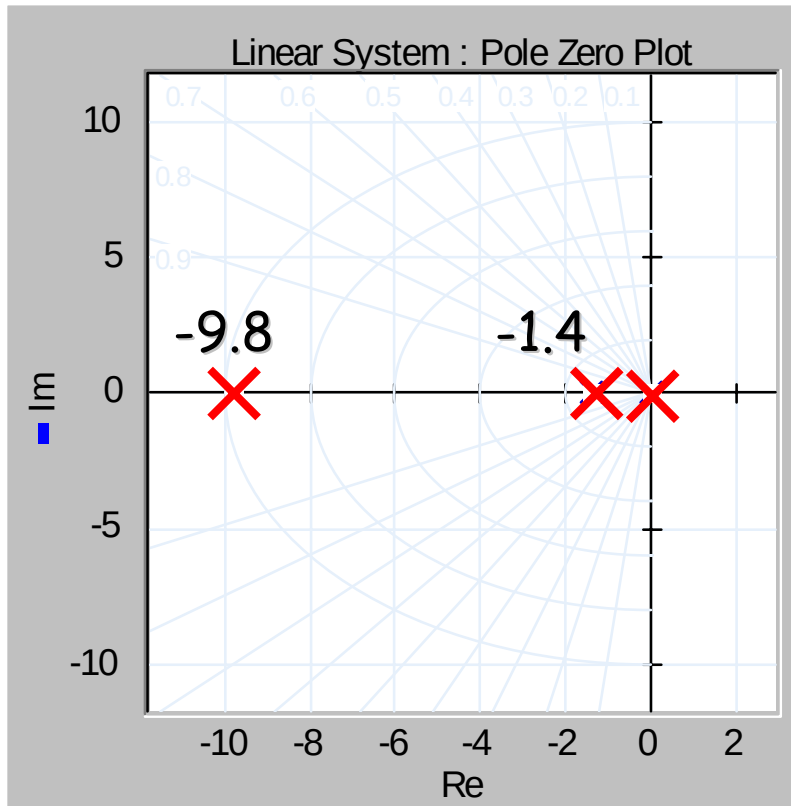
Linear System : Pole Zero Plot



Linear System : Bode Plot



After simplification (volt.)



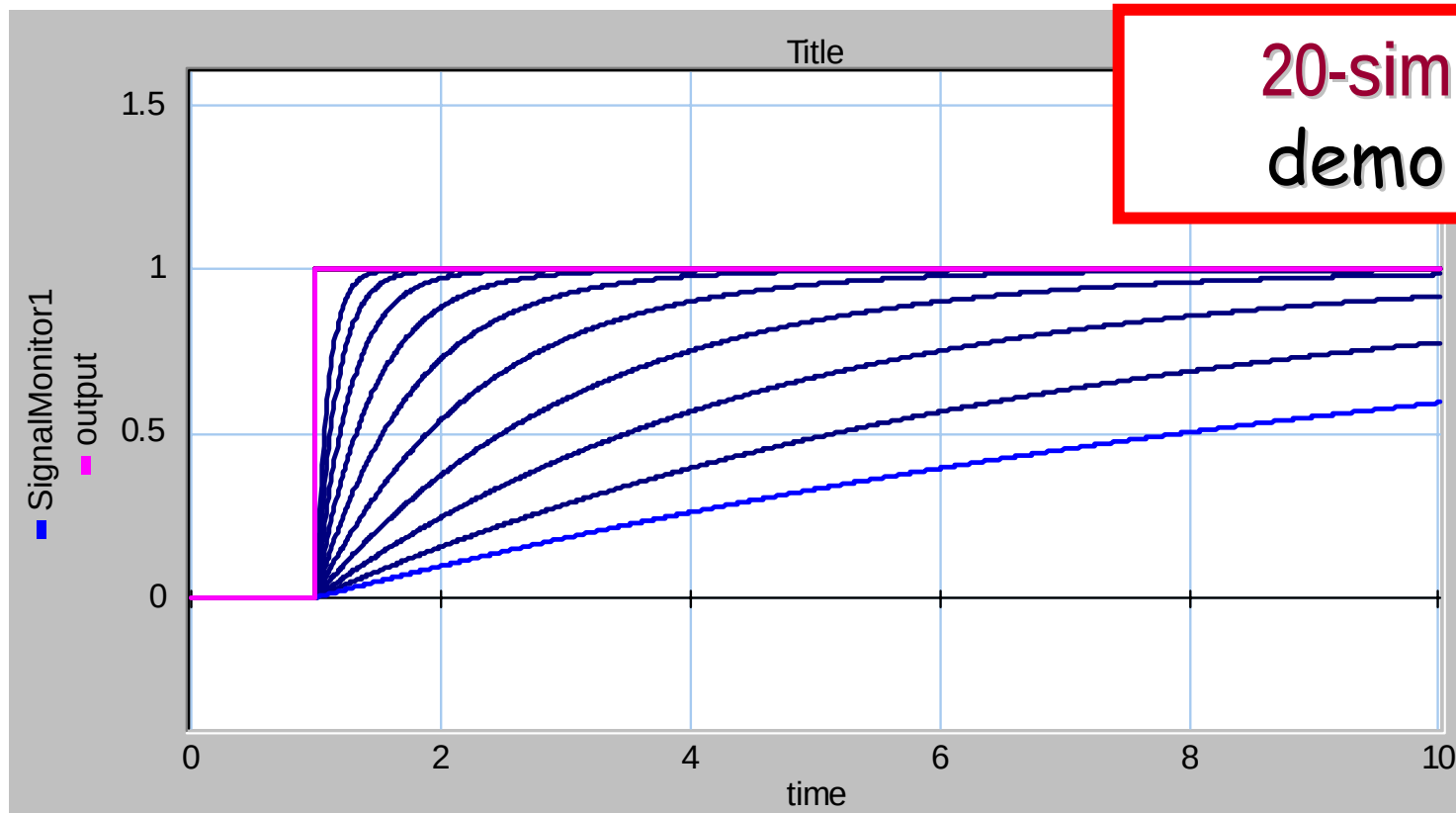
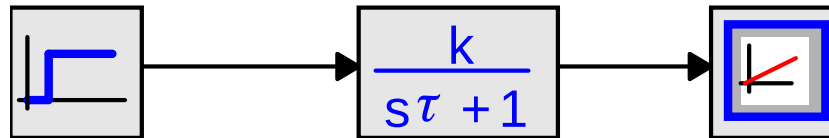
Relation s-domain

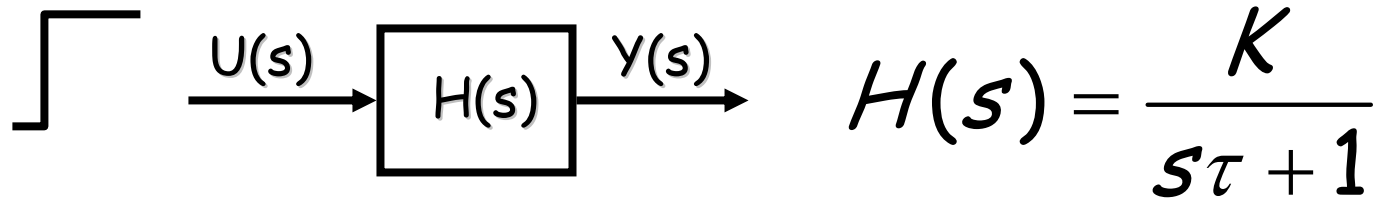


time domain

First-order system

$$0.1 < \tau < 10$$



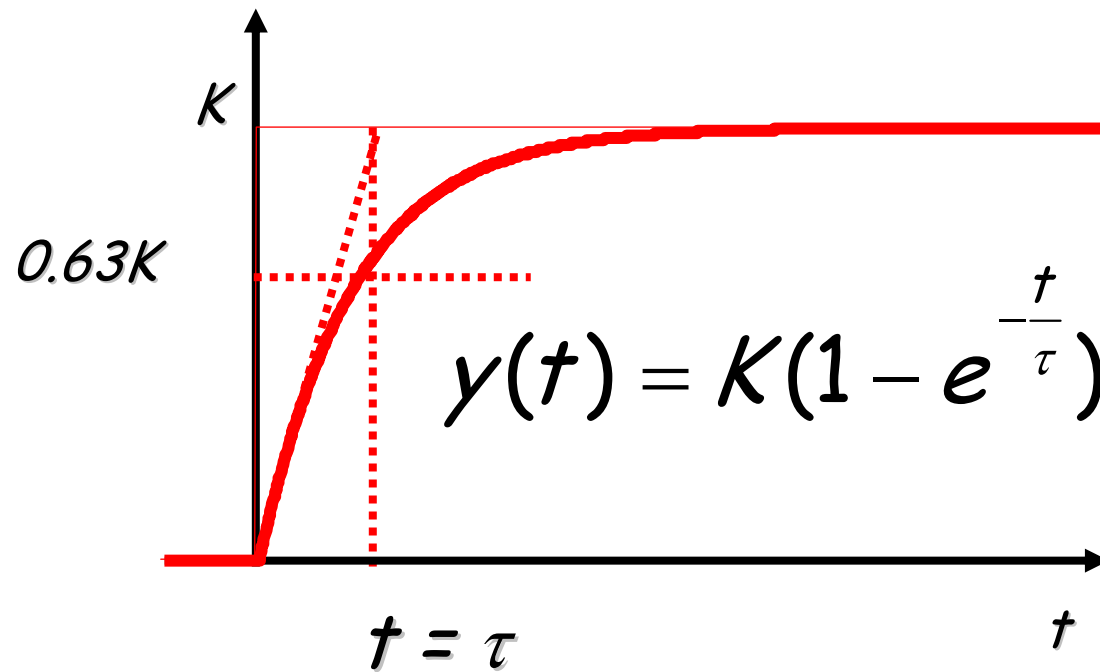
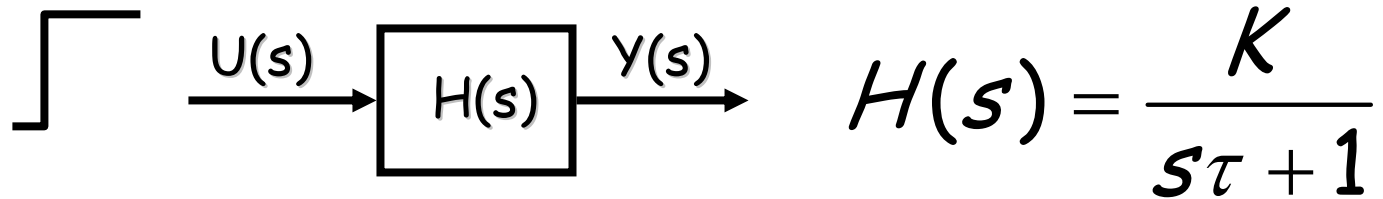


$$Y(s) = H(s) \frac{1}{s} = \frac{K}{s(s\tau + 1)}$$

$$= \frac{Ka}{s(s + a)} = K \left(\frac{1}{s} - \frac{1}{s + a} \right)$$

$$\rightarrow y(t) = K(1 - e^{-at}) = K(1 - e^{-\frac{t}{\tau}})$$

Solving the Differential Eq.



$$e^{-1} = 0.37$$

system gain

root locus gain

$$\frac{K}{s\tau + 1} = \frac{K / \tau}{s + 1/\tau} = \frac{Ka}{s + a} = \frac{K'}{s + a}$$

lowest power of $s = 1$

highest power of $s = 1$

A polynomial with the highest power of $s = 1$, is called a monic polynomial

system gain

root locus gain

$$\frac{K}{s(s\tau + 1)} = \frac{K / \tau}{s(s + 1/\tau)} = \frac{Ka}{s(s + a)} = \frac{K'}{s^2 + as}$$

lowest power of $s = 1$

highest power of $s = 1$

$$\frac{K}{s\tau + 1}$$

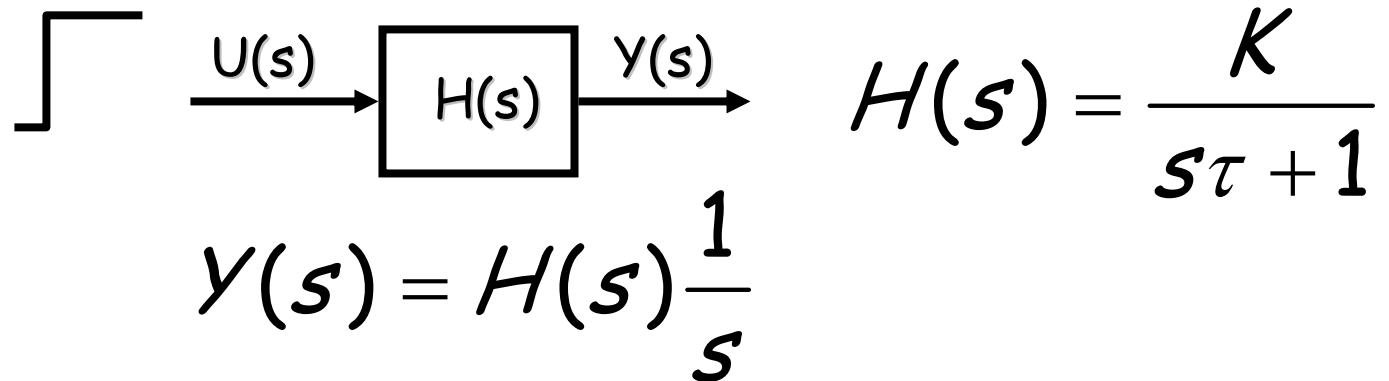
system gain = K

DC-gain = K

$$\frac{K}{s(s\tau + 1)}$$

system gain = K

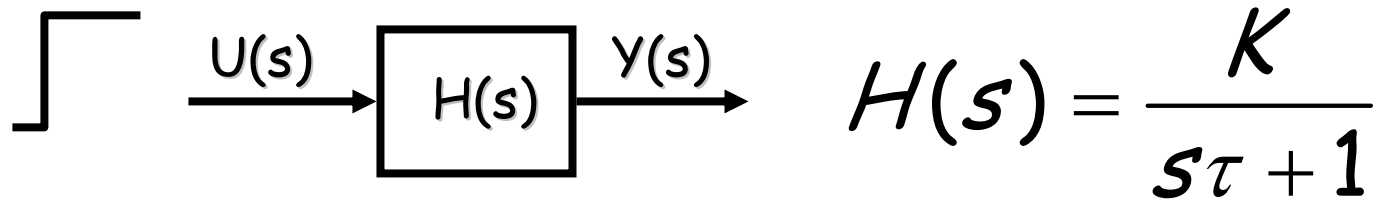
DC-gain = ∞



$$\lim_{t \rightarrow \infty} y(t) = \lim_{s \rightarrow 0} sY(s) = \lim_{s \rightarrow 0} H(s) =$$

$$\lim_{s \rightarrow 0} \frac{K}{s\tau + 1} = K$$

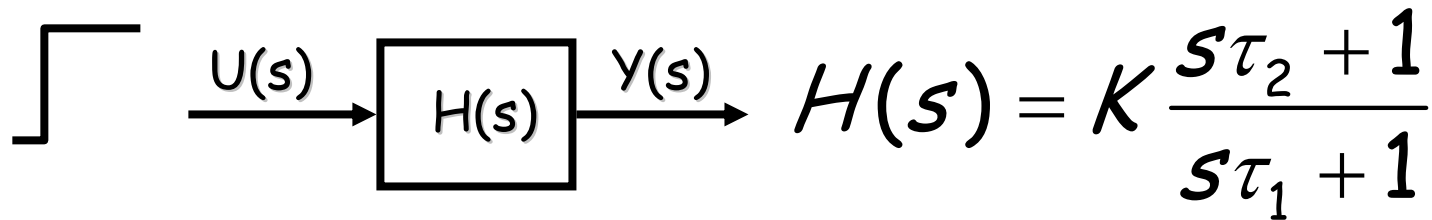
final value theorem



$$\lim_{t \rightarrow 0} y(t) = \lim_{s \rightarrow \infty} sY(s) = \lim_{s \rightarrow \infty} \frac{K}{s\tau + 1} = 0$$

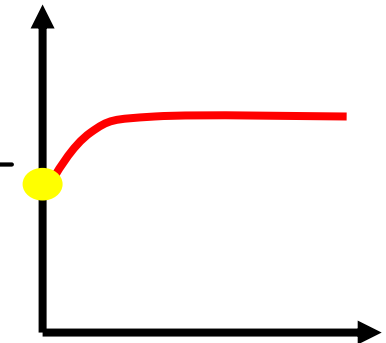
initial value theorem

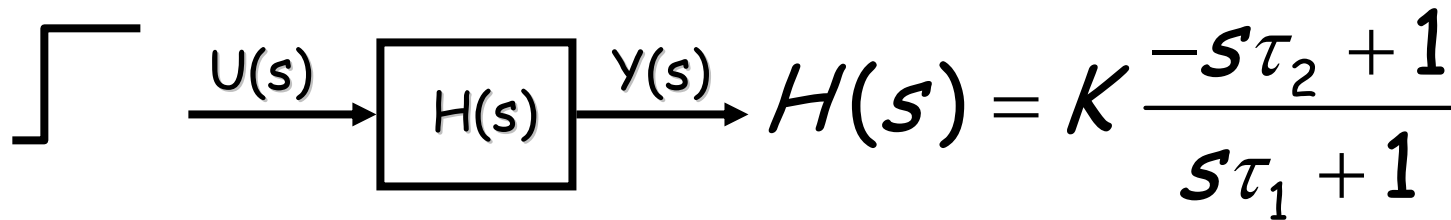
$$\lim_{t \rightarrow 0} \frac{d}{dt} (y(t)) = \lim_{s \rightarrow \infty} s(sY(s)) = \lim_{s \rightarrow \infty} \frac{Ks}{s\tau + 1} = \frac{K}{\tau}$$



$$\lim_{t \rightarrow 0} y(t) = \lim_{s \rightarrow \infty} sY(s) = K \frac{s\tau_2 + 1}{s\tau_1 + 1} = K \frac{\tau_2}{\tau_1}$$

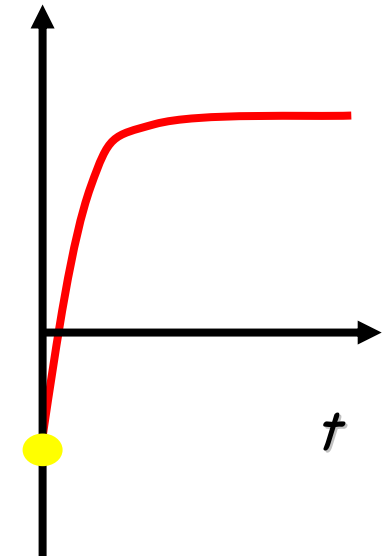
$$\lim_{t \rightarrow \infty} y(t) = \lim_{s \rightarrow 0} sY(s) = K \frac{s\tau_2 + 1}{s\tau_1 + 1} = K$$

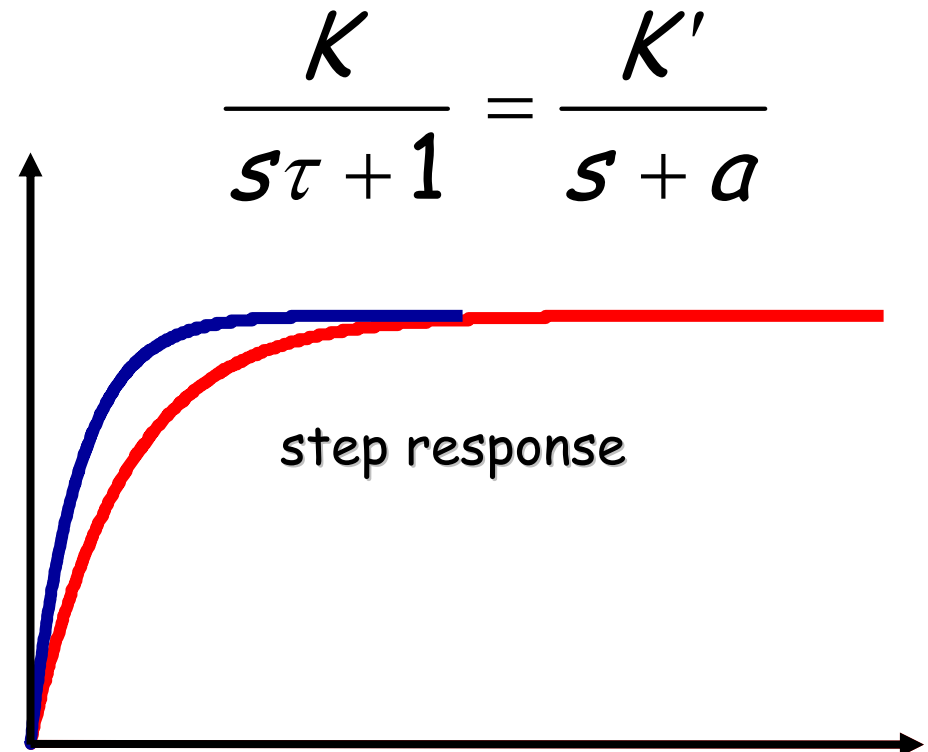
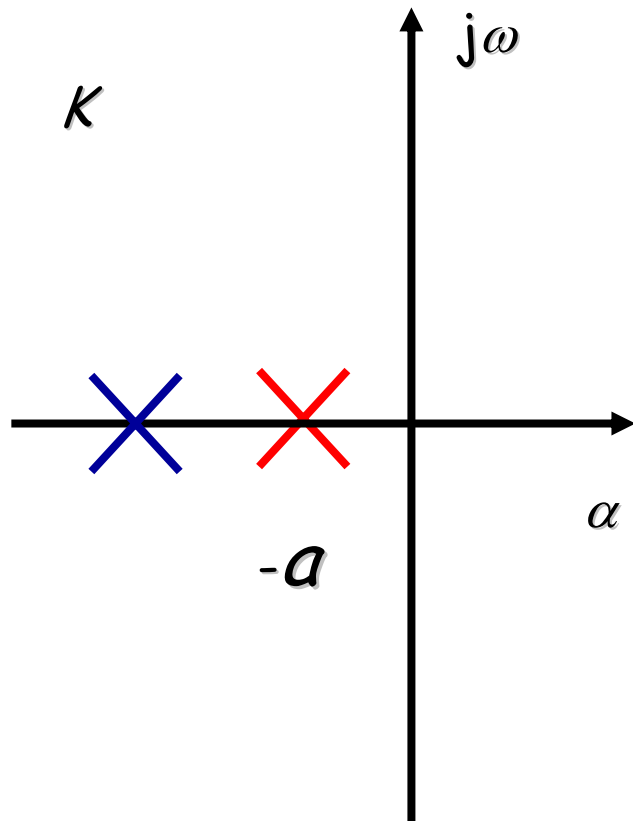


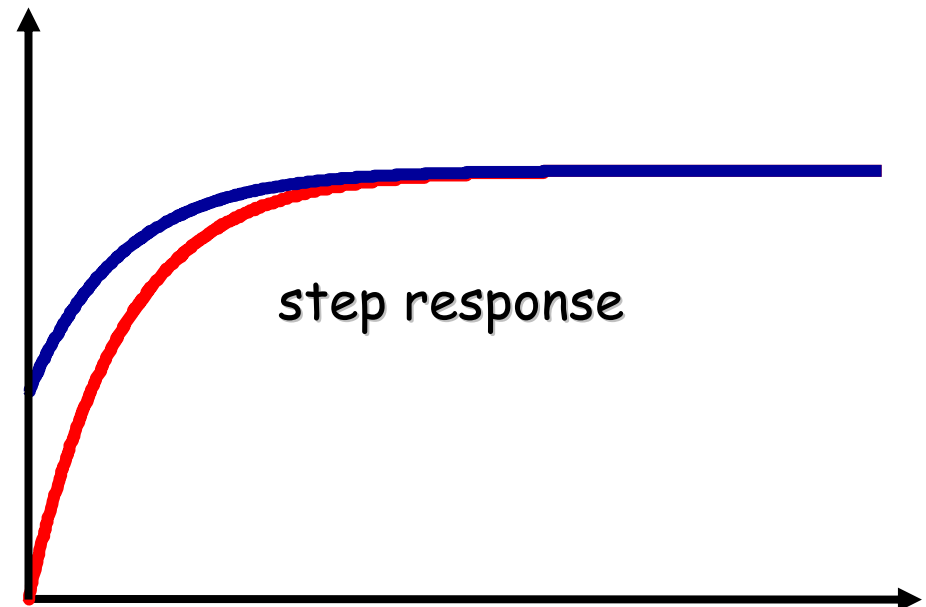
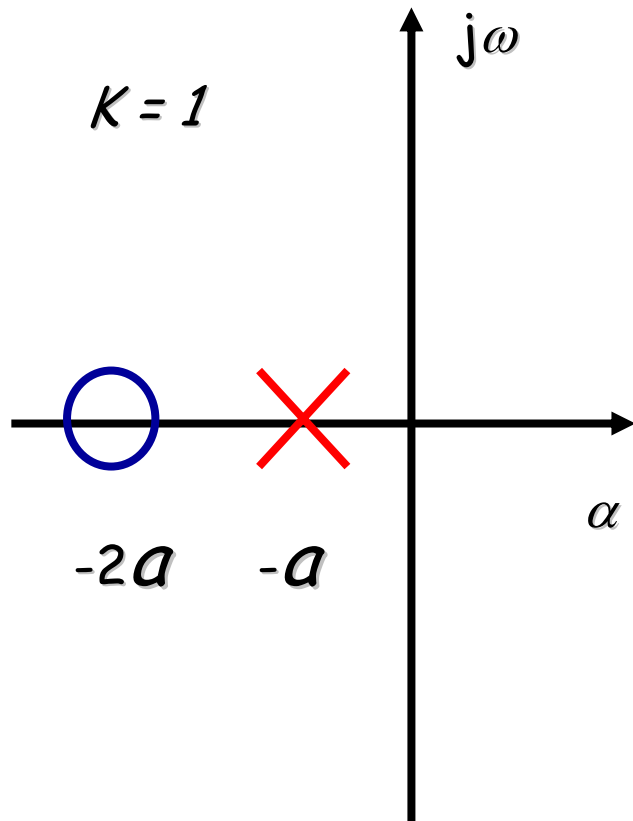


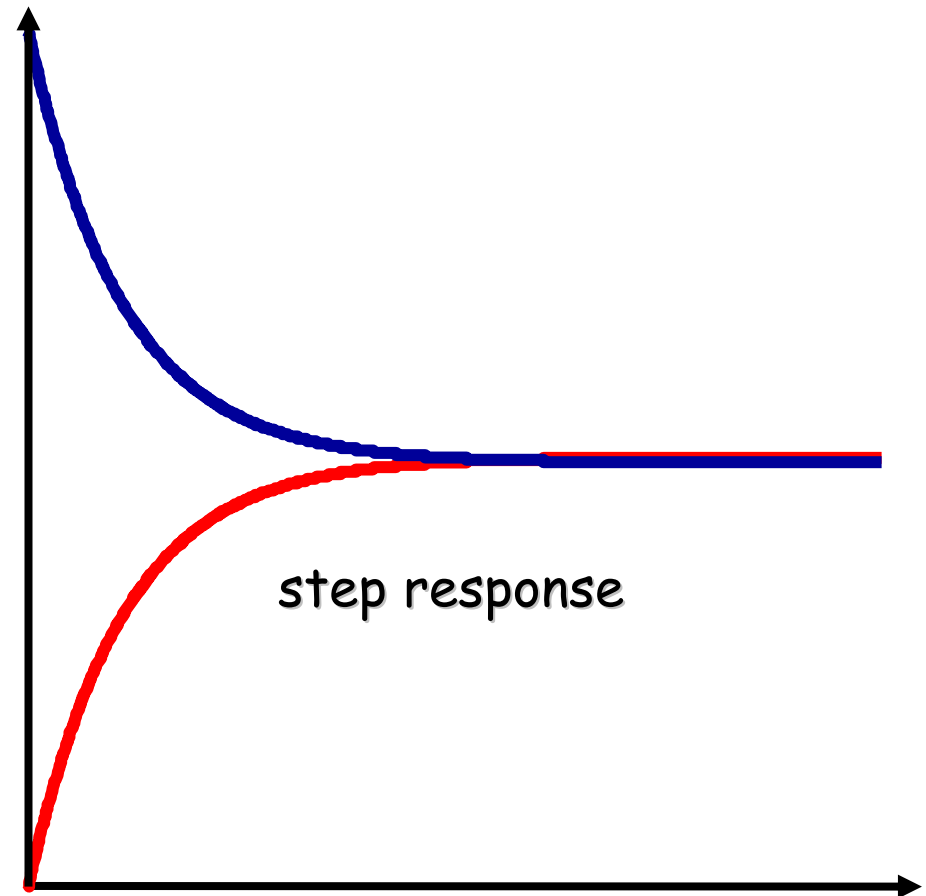
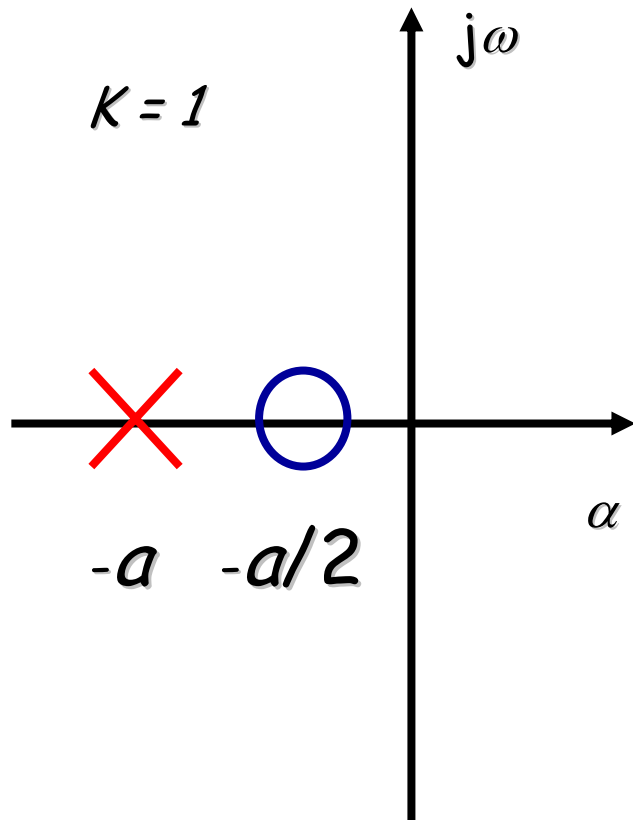
$$\lim_{t \rightarrow 0} y(t) = \lim_{s \rightarrow \infty} sY(s) = K \frac{-s\tau_2 + 1}{s\tau_1 + 1} = -K \frac{\tau_2}{\tau_1}$$

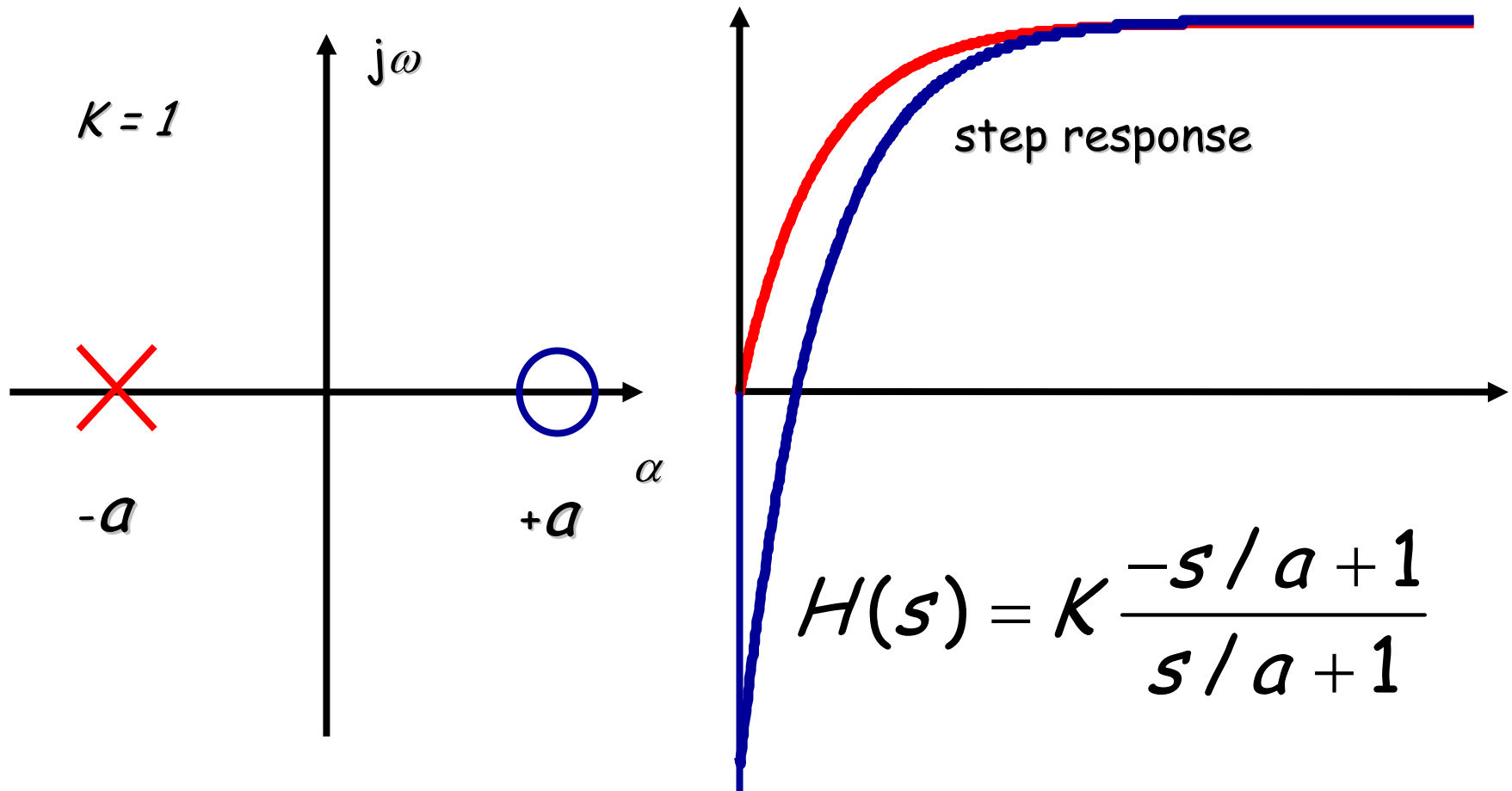
$$\lim_{t \rightarrow \infty} y(t) = \lim_{s \rightarrow 0} sY(s) = K \frac{-s\tau_2 + 1}{s\tau_1 + 1} = K$$

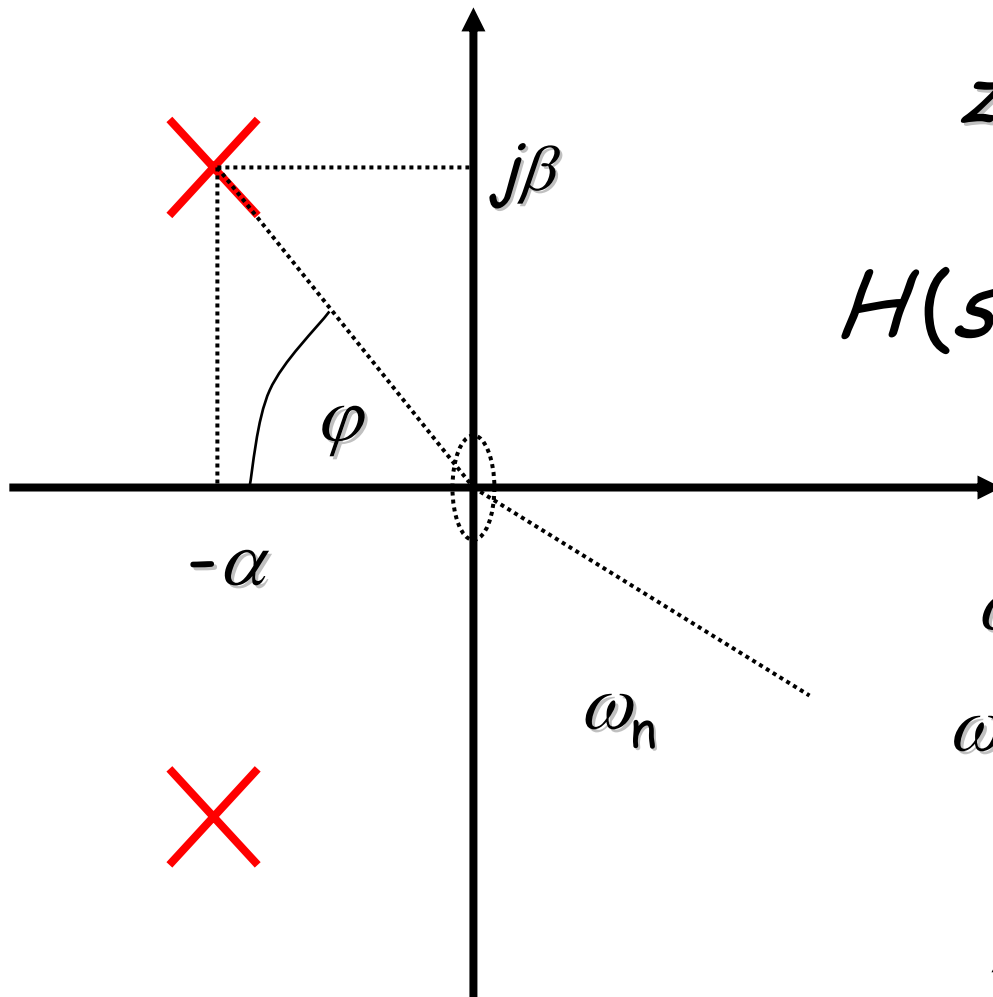












$z = \cos \varphi$ damping ratio

$$H(s) = \frac{\omega_n^2}{s^2 + 2z\omega_n s + \omega_n^2}$$

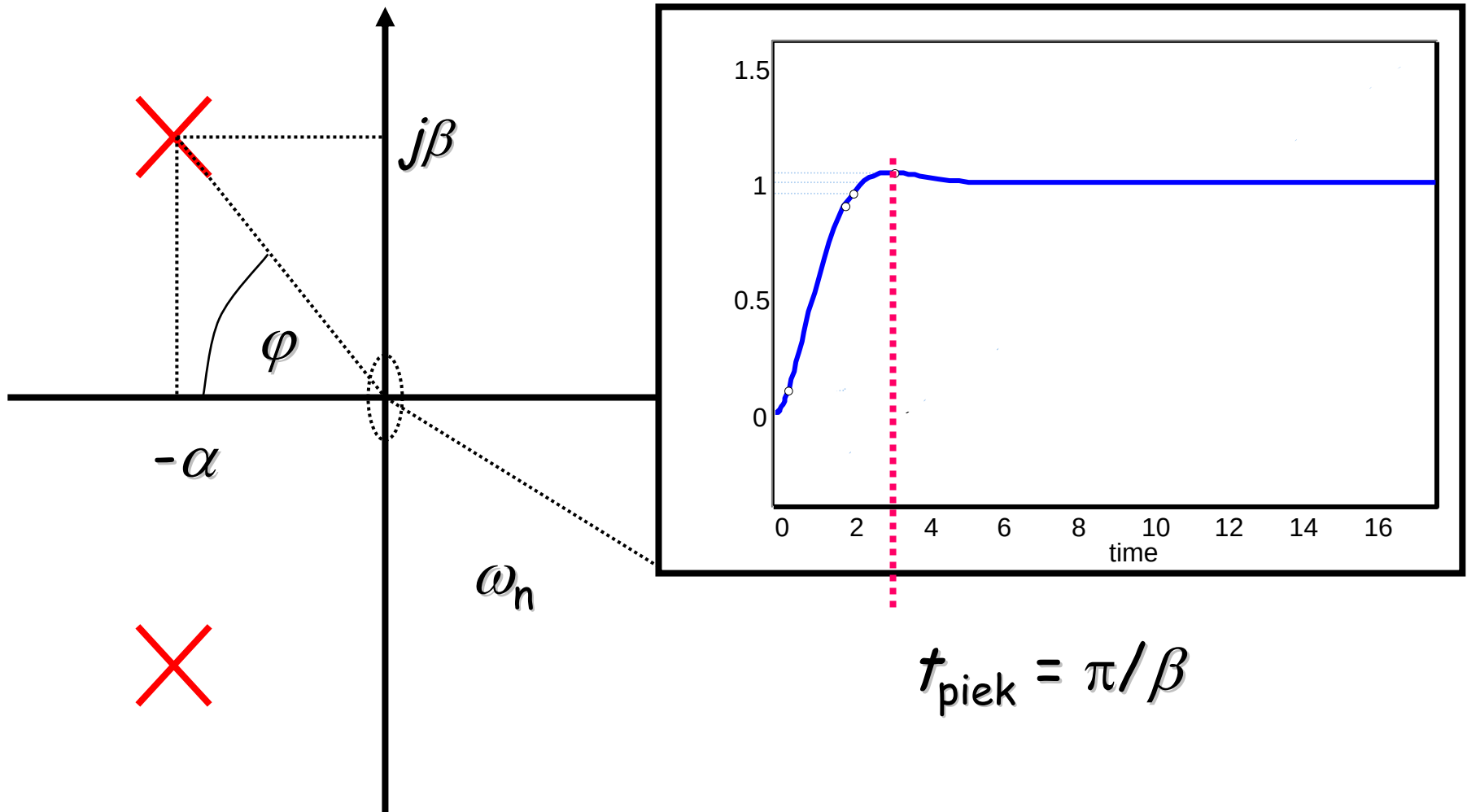
α = absolute damping

ω_n = natural frequency

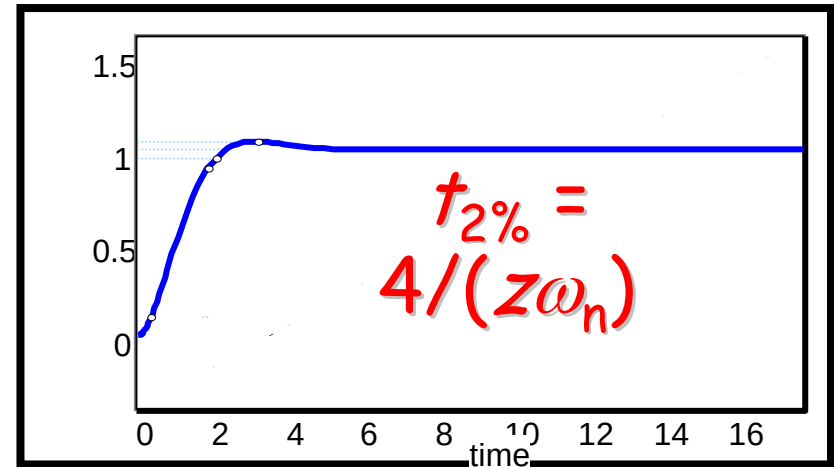
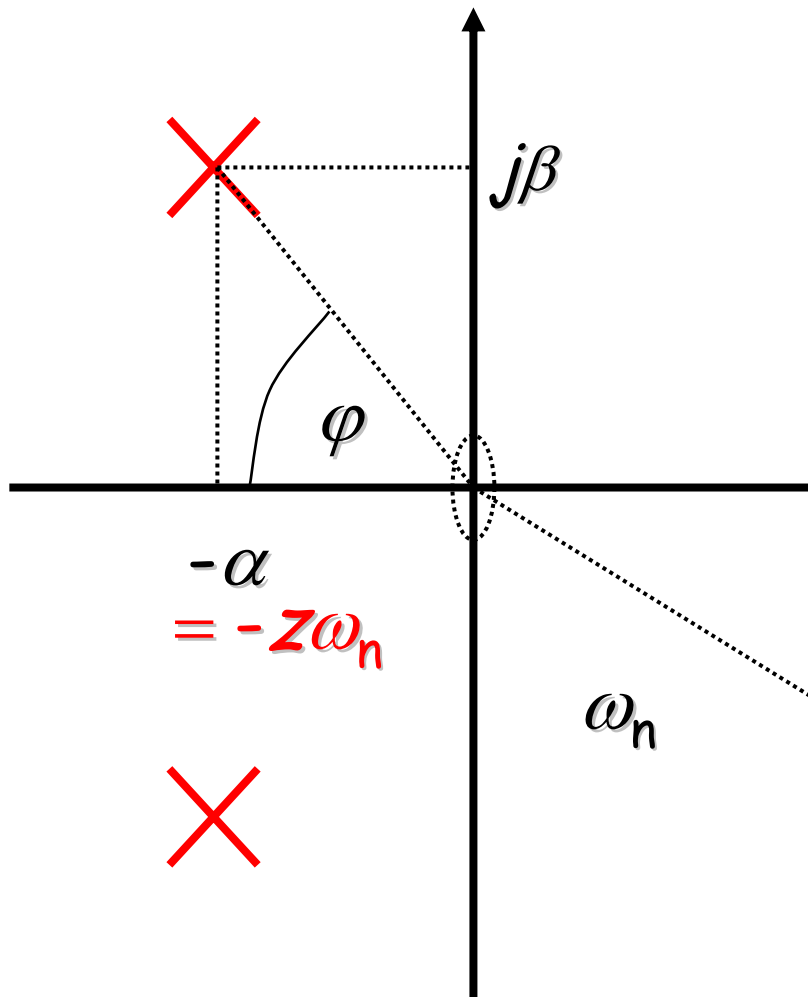
$$\alpha = \omega_n \cos(\varphi) = \omega_n z$$

$$\beta = \omega_n \sin(\varphi)$$

Second-order system

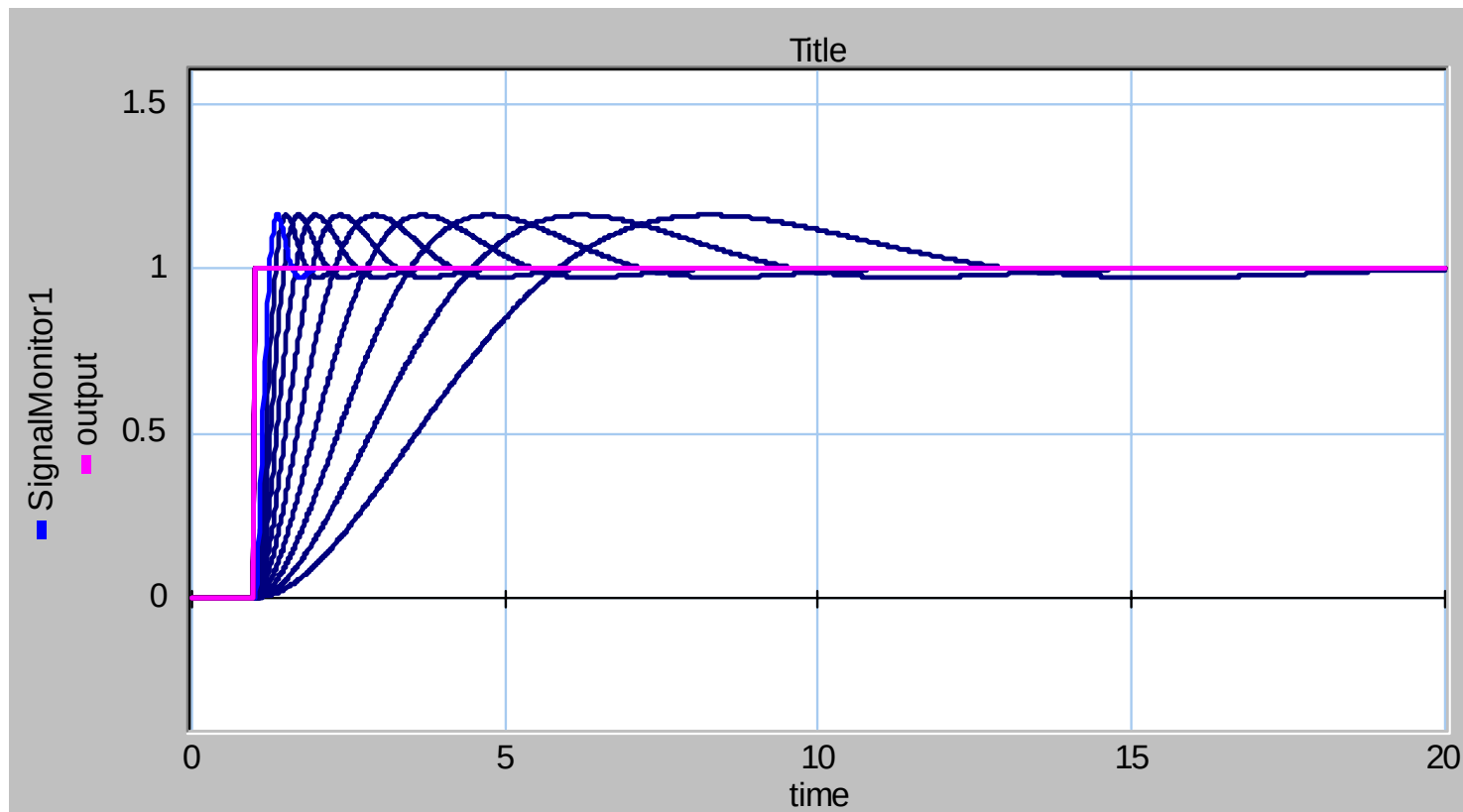
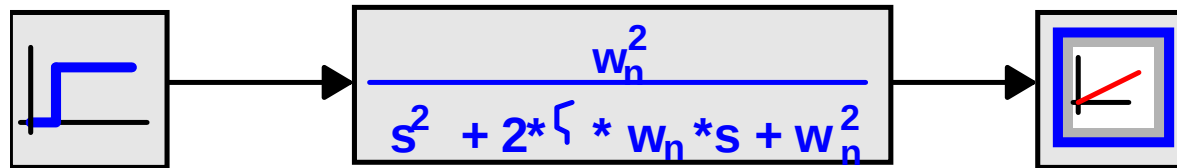


Second-order system

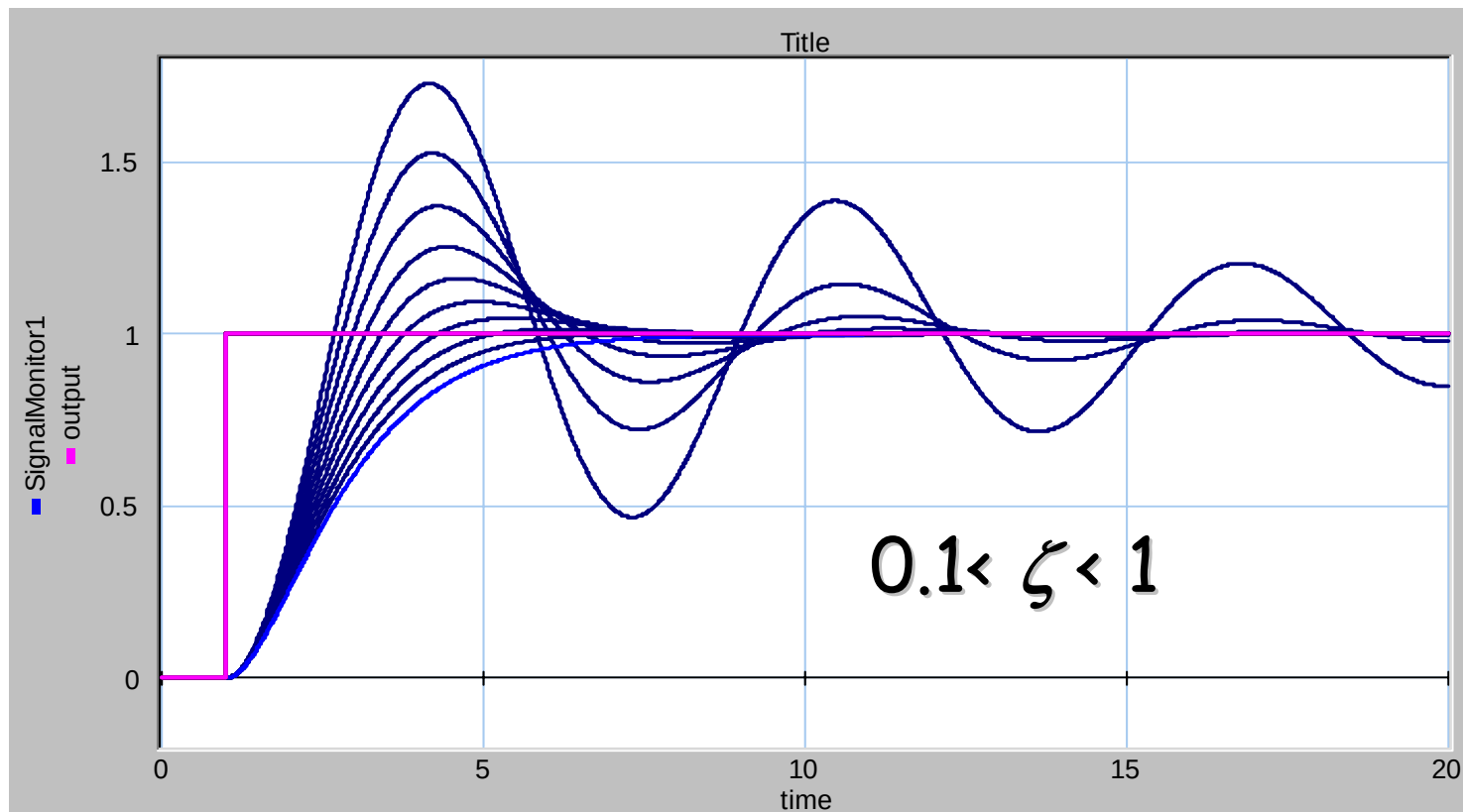
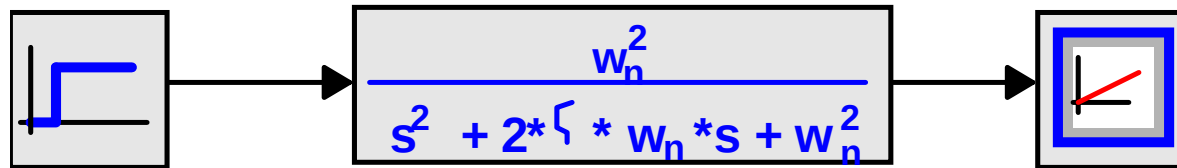


$t_{2\%}$ = time where
response remains
within 2% of final
value

Second-order system

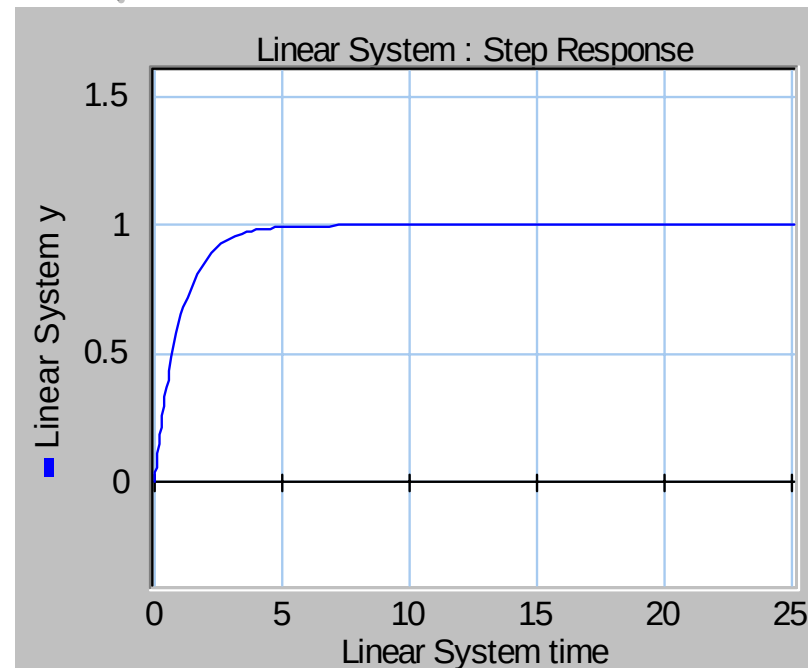
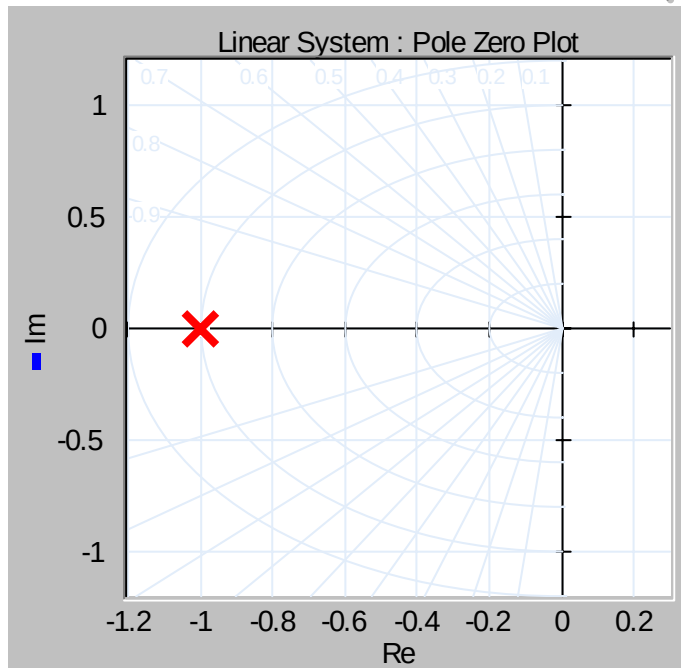


Second-order system



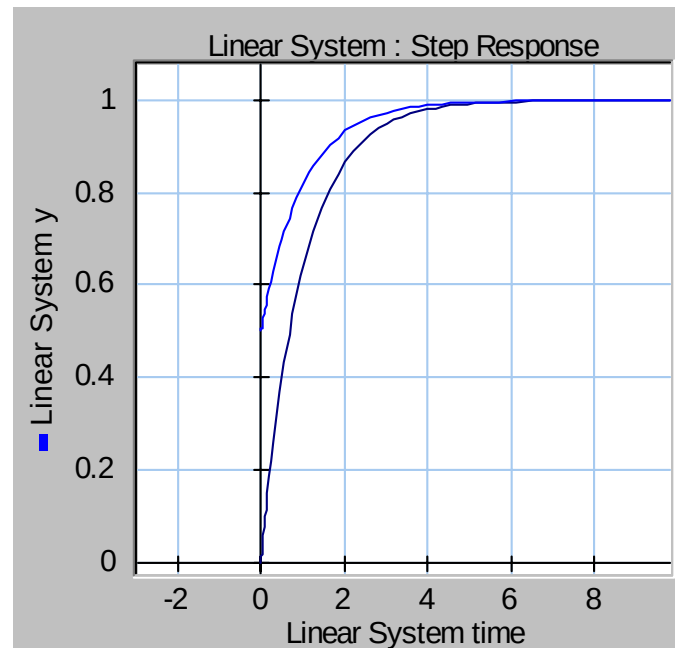
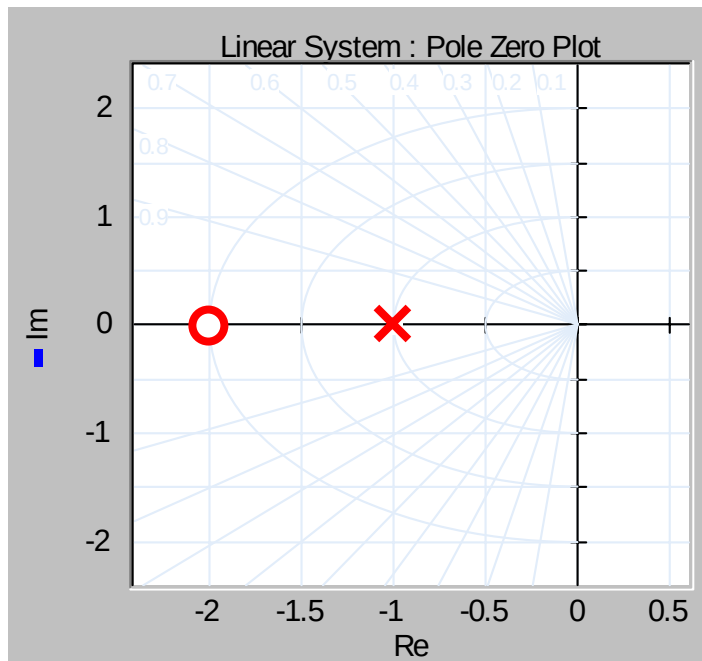
Exercise (1)

Use the linear systems editor of **20-sim** to simulate a first-order system and investigate the influence of a varying gain and time constant on the step response



Exercise (2)

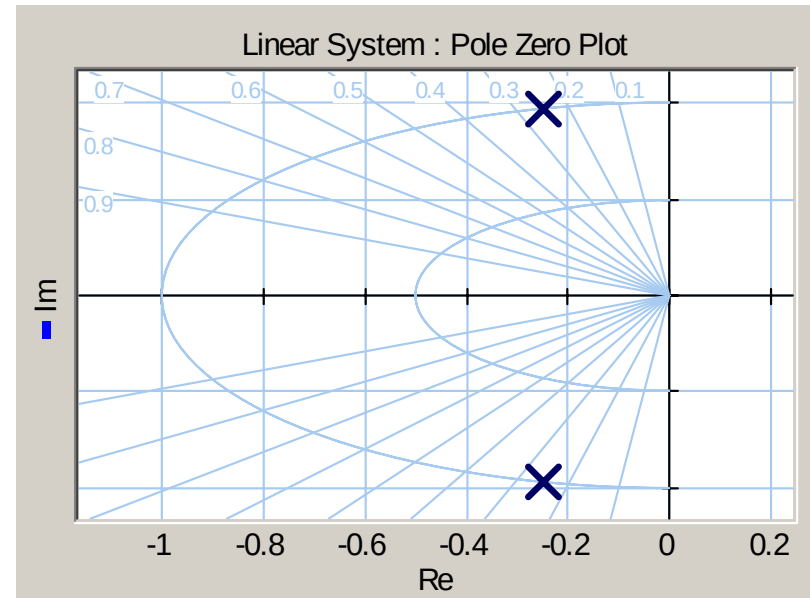
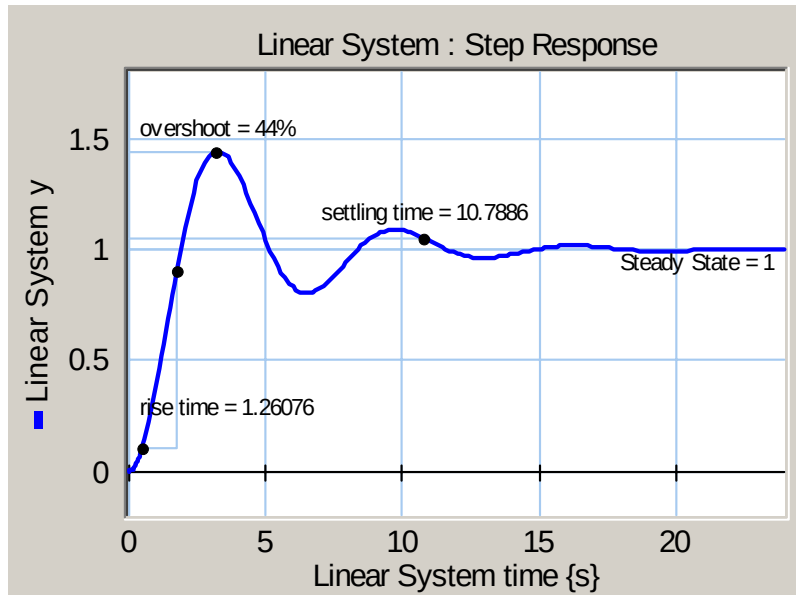
Do the same for a first-order system with a zero and investigate the influence of a varying zero on the step response



Exercise (3)

Investigate the influence of varying ω_n en z for a second-order system on the step response.

Investigate the effect of an extra pole or zero



Check the relation between peak time and settling time and the location of the poles of a second order system.

Generate the responses with **20-sim**